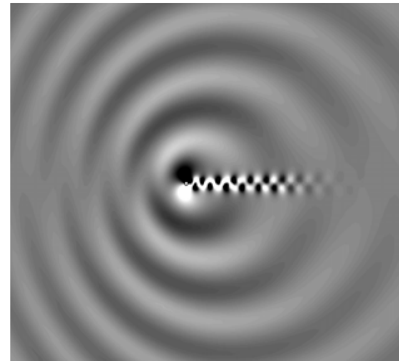


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**Computational Aeroacoustics**  
*for*  
**Low Mach Number Flows (I)**



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Korea University  
Department of Mechanical Engineering  
Seoul, Korea

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# Contents

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- I. Perturbed Compressible Equations (*PCE*) –  $M \ll 1$
  - II. Computational Issues of *PCE* – *efficiency & accuracy*
  - III. Linearized Perturbed Compressible Equations (*LPCE*) –  $M \ll \ll 1$
- 
- IV. 3D Flow past a Circular Cylinder –  $Re_D = 46000$  &  $M = 0.21$
  - V. Phonation Aero-acoustics in Vocal Tract
  - VI. Concluding Remarks



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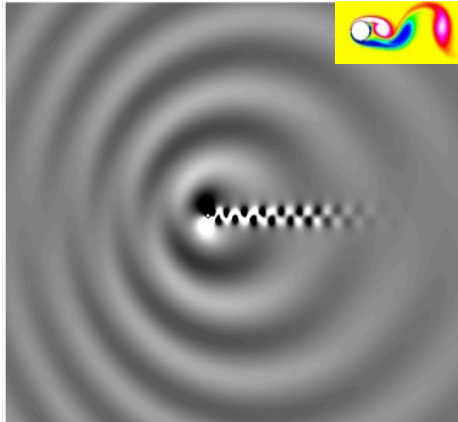
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I. Perturbed Compressible Equations (*PCE*) –  $M \ll 1$

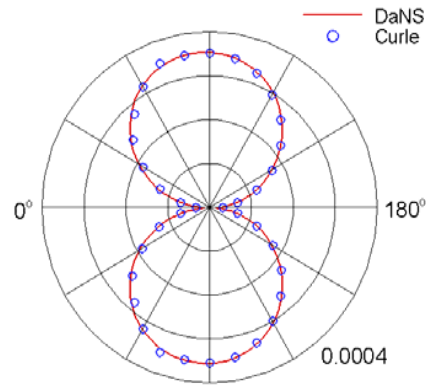


## DNS – cylinder dipole tone (laminar)

$$\Delta p' = p - \bar{p}$$



$Re_D = 200, M_\infty = 0.3$



@  $r = 60D$

- DNS (compressible NS)
- 6<sup>th</sup>-order compact scheme  
4<sup>th</sup>-order RK
- Non-reflecting BC  
(ETA, *Energy Transfer & Annihilating*) )

- $\lambda_a / \lambda_h \propto 1/M_\infty$
- Large disparity between flow and acoustic scales.
- DNS becomes very costly.

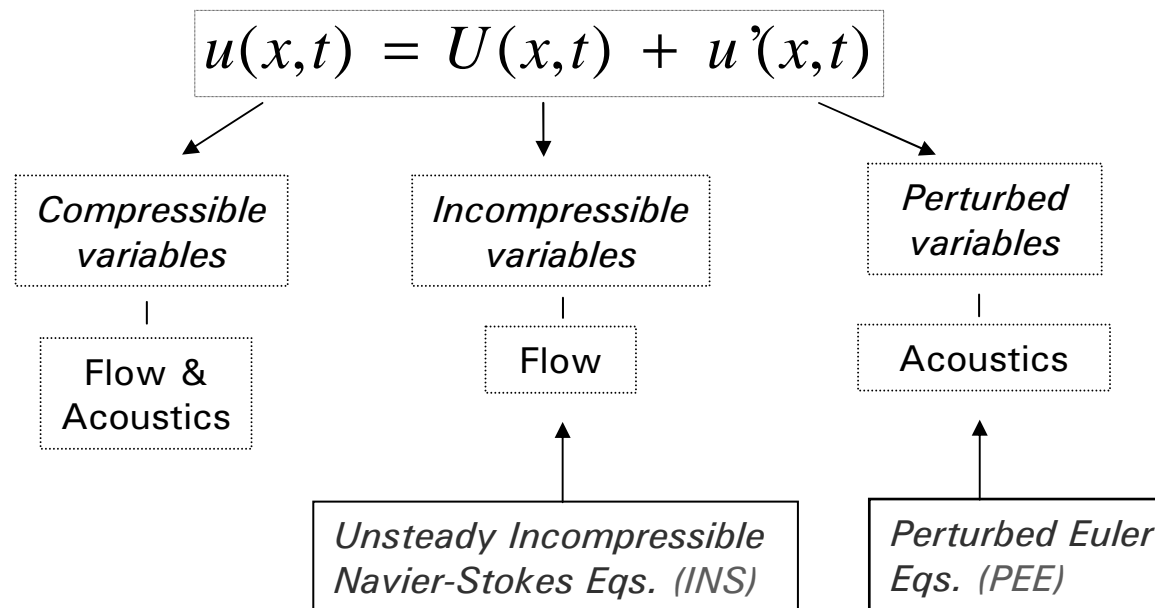


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## CFD / CAA Hybrid method

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- Hydrodynamic-Acoustic splitting concept
  - *Hardin & Pope (1992, 1995)*
  - *Shen & Sorenson (1999, 2001)*
  - *Slimon et al. (1999, 2000)*



# Perturbed Euler Eqs. (PEE)

$$\rho = \rho_0 + \rho_1 + \rho', \quad u = U + u', \quad p = P + p'$$

$$\text{CNS} - \text{INS} = \text{PEE}$$

neglecting the viscous terms

- $\frac{\partial \rho'}{\partial t} + \frac{\partial}{\partial x_i}(\rho u'_i + \rho' U_i) = -\frac{\partial \rho_1}{\partial t} - U_i \frac{\partial \rho_1}{\partial x_i}$
- $\frac{\partial}{\partial t}(\rho u'_i + \rho' U_i) + \frac{\partial}{\partial x_j} \left[ u_i(\rho u'_i + \rho' U_i) + (\rho_0 + \rho_1) U_i u'_j + p' \delta_{ij} \right] = -\frac{\partial(\rho_1 U_i)}{\partial t} - U_j \frac{\partial(\rho_1 U_i)}{\partial x_j}$

*Shen & Sorenson (1999)*

*Slimon et al. (2000)*

- $\rho_1 = 0$

- $\frac{\partial p'}{\partial t} - a^2 \frac{\partial \rho'}{\partial t} = -\frac{\partial P}{\partial t}$

- $\frac{D\rho_1}{Dt} = \frac{1}{a^2} \frac{DP}{Dt}$

- $\frac{Dp'}{Dt} - a^2 \frac{D\rho'}{Dt} = a^2 \left( \frac{\gamma-1}{2} \right) \frac{D\rho_1}{Dt}$

- Slip condition at the wall

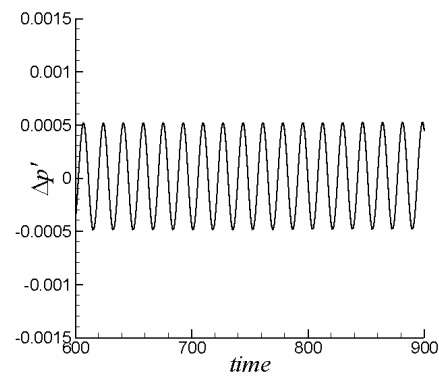
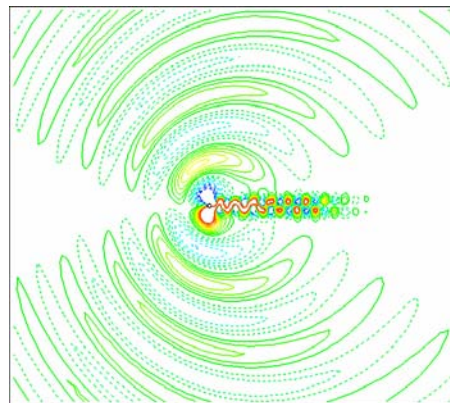


# Validation – *INS* + *PEE*

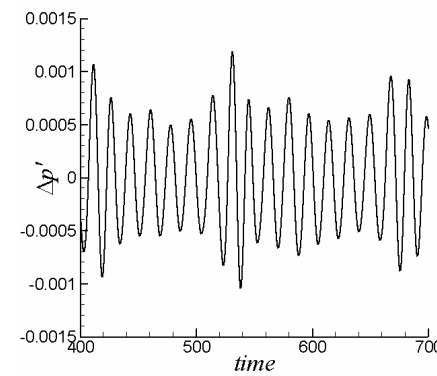
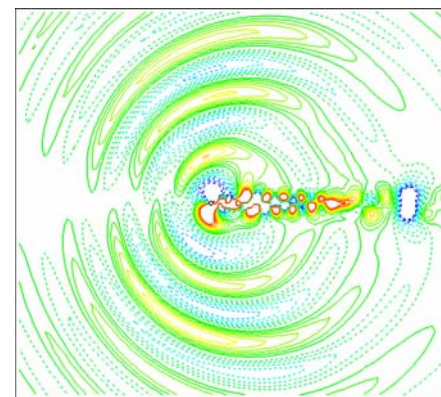
$$\Delta p' = p - \bar{p}$$

- *INS*: iterative fractional step method, 6<sup>th</sup> order for Mom. eq., 2<sup>nd</sup> order for Poisson eq., RK-3.
- Same grid for hydro & acoustic calcs. for excluding any interpolation errors.

*DNS*



*Shen & Sorenson (PEE)*



$\Delta p'$  time history at  $r = 60D$  and  $\Theta = 90^\circ$

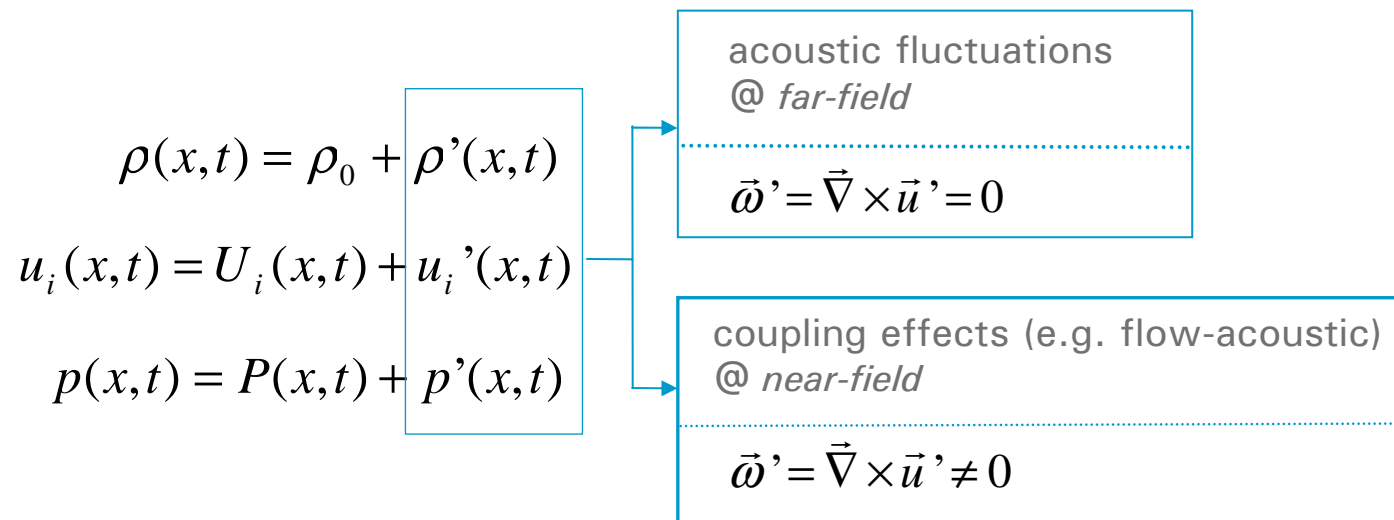


# Derivation of *Perturbed Compressible Eqs. (PCE)*

- **Seo & Moon** (2003, AIAA paper 2003-3270)

*"A Hybrid Method for Aeroacoustic Noise Prediction of Wall-bounded Shear Flows at Low Mach Numbers", will appear in AIAA J. 2005.*

- Understandings from the *false* solution of *PEE*



A coupling effect becomes important near the wall, when dealing with the noise generated by wall-bounded shear flows, and *it is associated with*  $\vec{\omega}'$ .





# Perturbed vorticity transport eq. (2D)

$$w = \Omega + w'$$

$$\frac{\partial \Omega}{\partial t} + (\vec{U} \cdot \vec{\nabla}) \Omega = \vec{\nabla} \times \vec{F}_{vis, I}$$

Incompressible

$$\begin{aligned} \frac{\partial w}{\partial t} + (\vec{u} \cdot \vec{\nabla}) w = & -w (\vec{\nabla} \cdot \vec{u}) \\ & - \vec{\nabla} \times \left( \frac{1}{\rho} \vec{\nabla} p \right) + \vec{\nabla} \times \vec{F}_{vis, C} \end{aligned}$$

Compressible

$$\frac{D(w'/\rho)}{Dt} =$$

Perturbed

$$\begin{aligned} & -\frac{1}{\rho} [(\vec{u}' \cdot \vec{\nabla}) \Omega + \Omega (\vec{\nabla} \cdot \vec{u}')] \\ & + \frac{1}{\rho} \vec{\nabla} \times \vec{F}'_{vis} + \frac{1}{\rho} (\vec{\nabla} T \times \vec{\nabla} s) \end{aligned}$$

Production by interaction between incompressible vorticity & perturbed velocities  $\Omega$  &  $\vec{u}'$  (in near-field)

Diffusion by perturbed viscous stresses (near field)

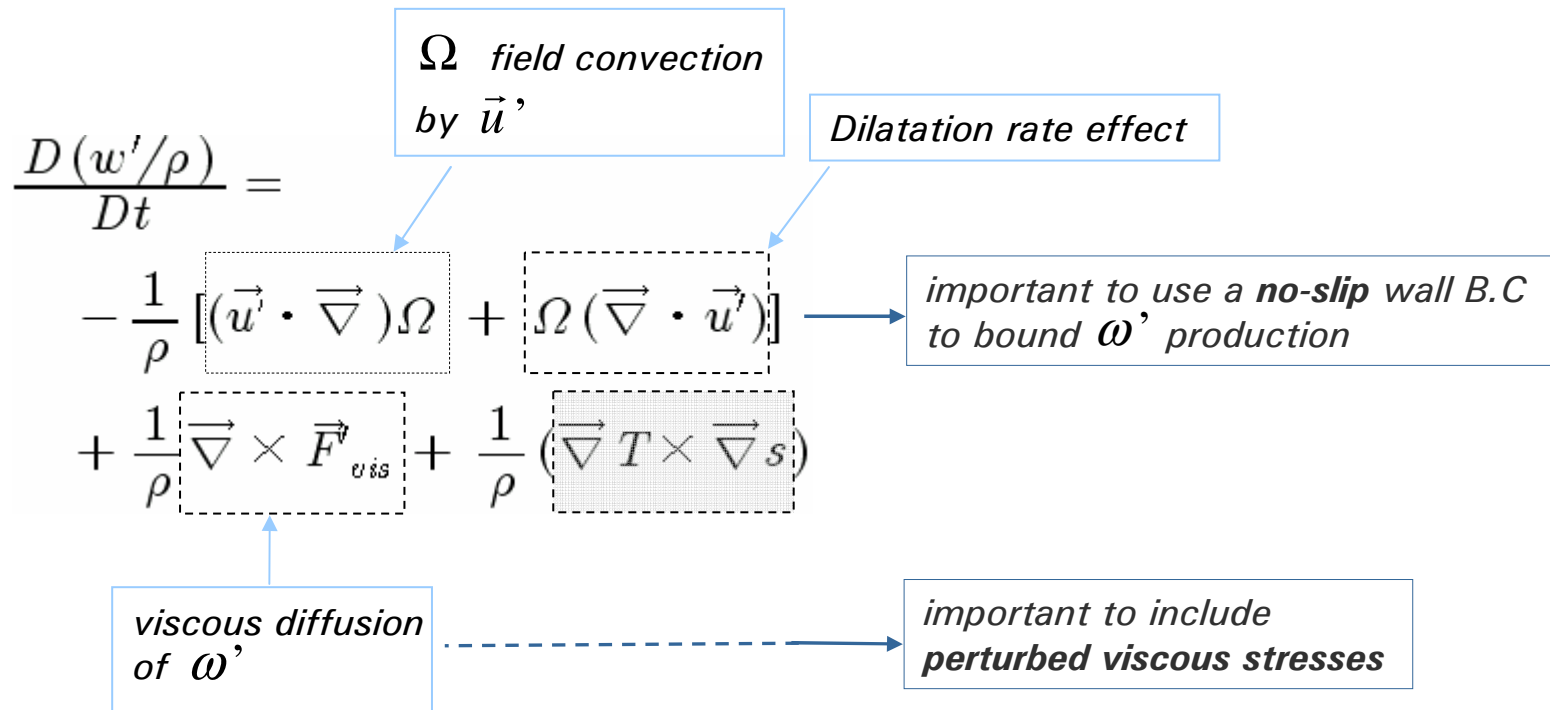
Production by entropy & temperature gradients

$$\vec{F}'_{vis} = \vec{F}_{vis, C} - \vec{F}_{vis, I}$$



# Source terms of perturbed vorticity

In *PEE*,  $u'$  is not zero at the wall, where  $\Omega$  computed by INS is most intense. So full interactions occur there, contributing to produce  $\omega'$  without proper diffusion.



# Perturbed Compressible Eqs. (PCE)

$$\rho = \rho_0 + \rho', \quad u = U + u', \quad p = P + p'$$

$$\text{CNS} - \text{INS} = \text{PCE}$$

including the perturbed viscous terms

- $\frac{\partial \rho'}{\partial t} + \frac{\partial}{\partial x_i} (\rho u'_i + \rho' U_i) = 0$

- $\frac{\partial}{\partial t} (\rho u'_i + \rho' U_i) + \frac{\partial}{\partial x_j} \left[ u_i (\rho u'_i + \rho' U_i) + \rho_0 U_i u'_j + p' \delta_{ij} \right] = \frac{\partial \tau'_{ij}}{\partial x_j}$ ,

where  $\tau'_{ij} = \mu_0 \left( \partial_j u'_i + \partial_i u'_j - \frac{2}{3} \delta_{ij} \partial_k u'_k \right)$  ← obtained by assuming  $\mu = \mu_0$  for  $M < 1$

- $\frac{\partial p'}{\partial t} + u_j \frac{\partial p'}{\partial x_j} + \gamma p \frac{\partial u'_j}{\partial x_j} = - \frac{DP}{Dt} + (\gamma - 1) \left( \phi - \frac{\partial q_j}{\partial x_j} \right)$

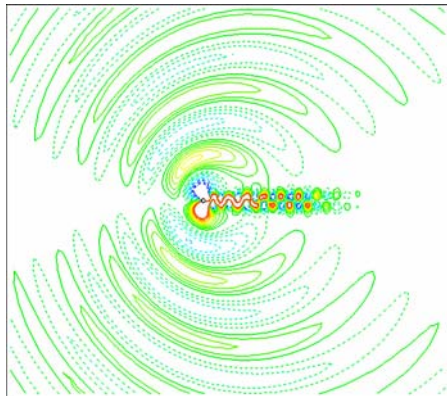
• No-slip condition at the wall



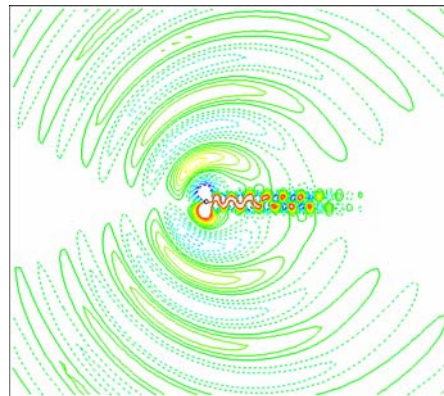
# Validation – *INS* + *PCE*

$$\Delta p' = p - \bar{p}$$

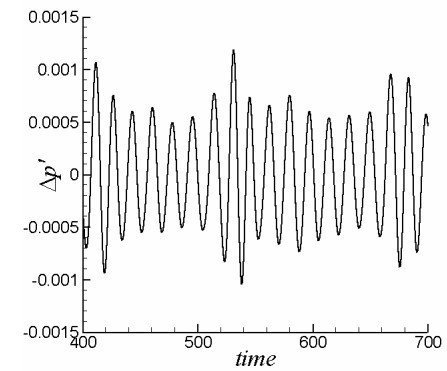
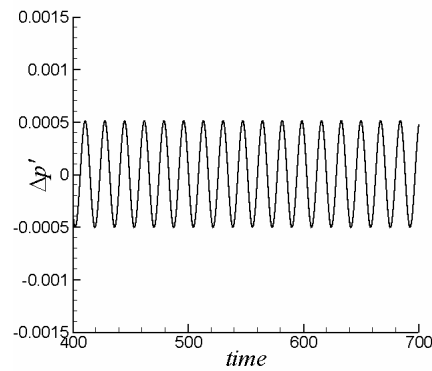
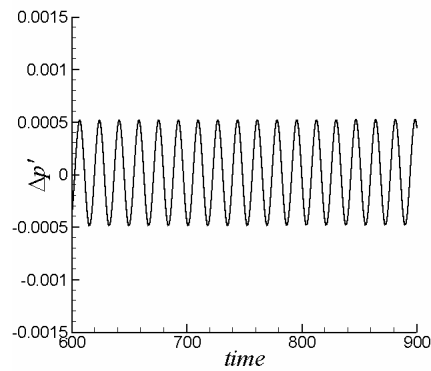
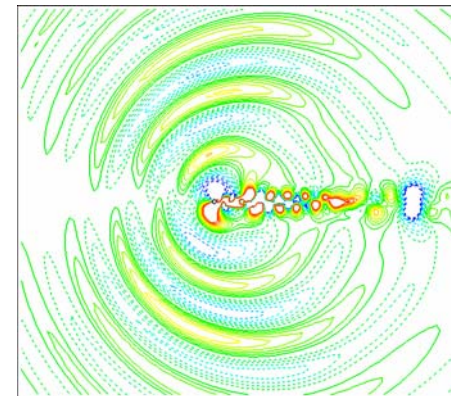
*DNS*



*Present (PCE)*



*Shen & Sorenson (PEE)*

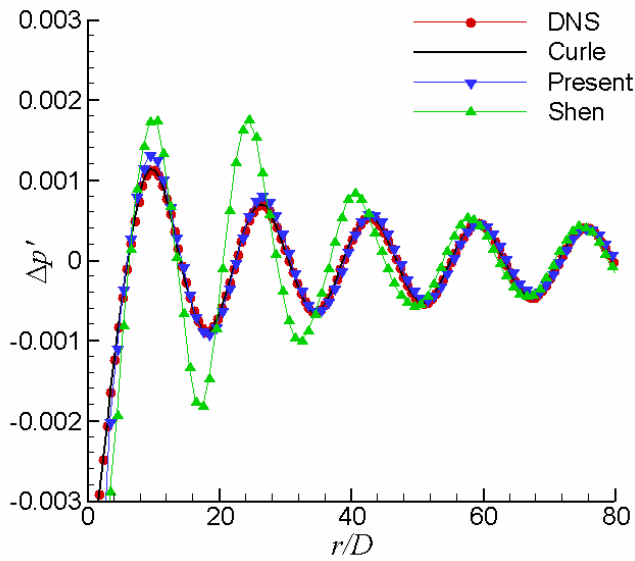


$\Delta p'$  time history at  $r = 60D$  and  $\Theta = 90^\circ$

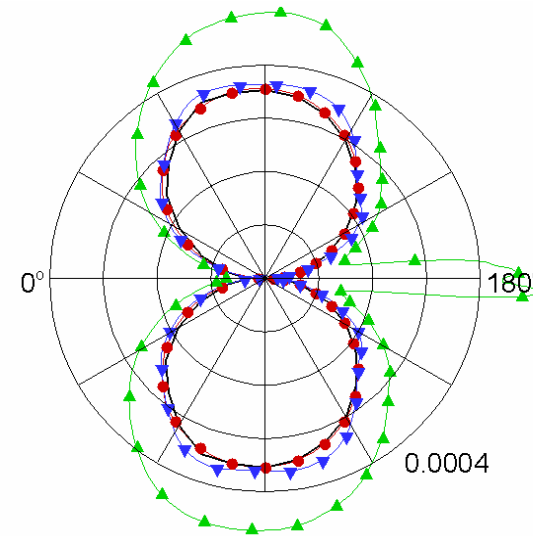


# Validation – pressure fluctuations

$$\Delta p' = p - \bar{p}$$



acoustic pressure wave at  $x=0$

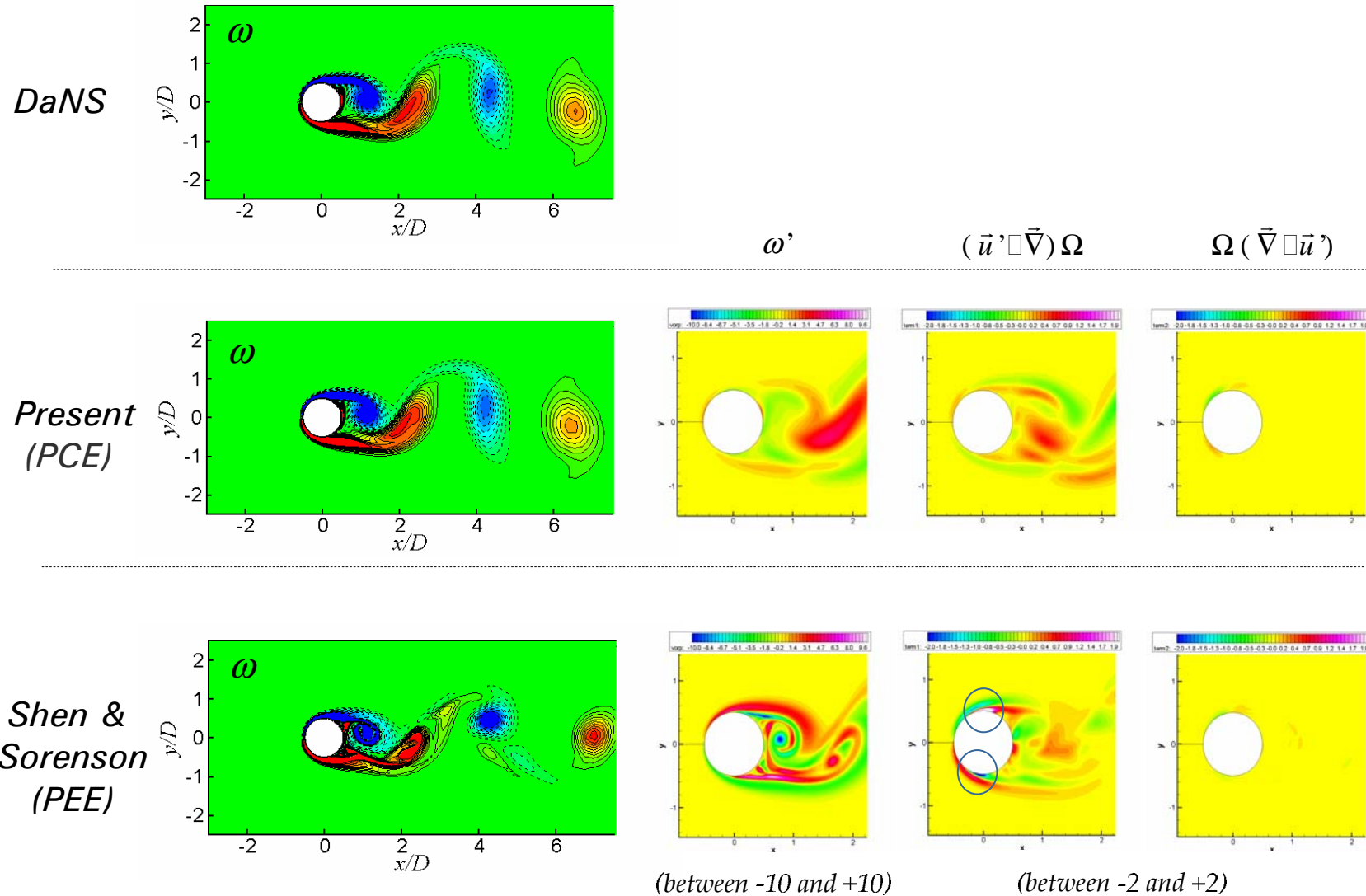


directivity of  $\Delta p'_{rms}$  ( $r=60D$ )



# Validation – vorticity fields

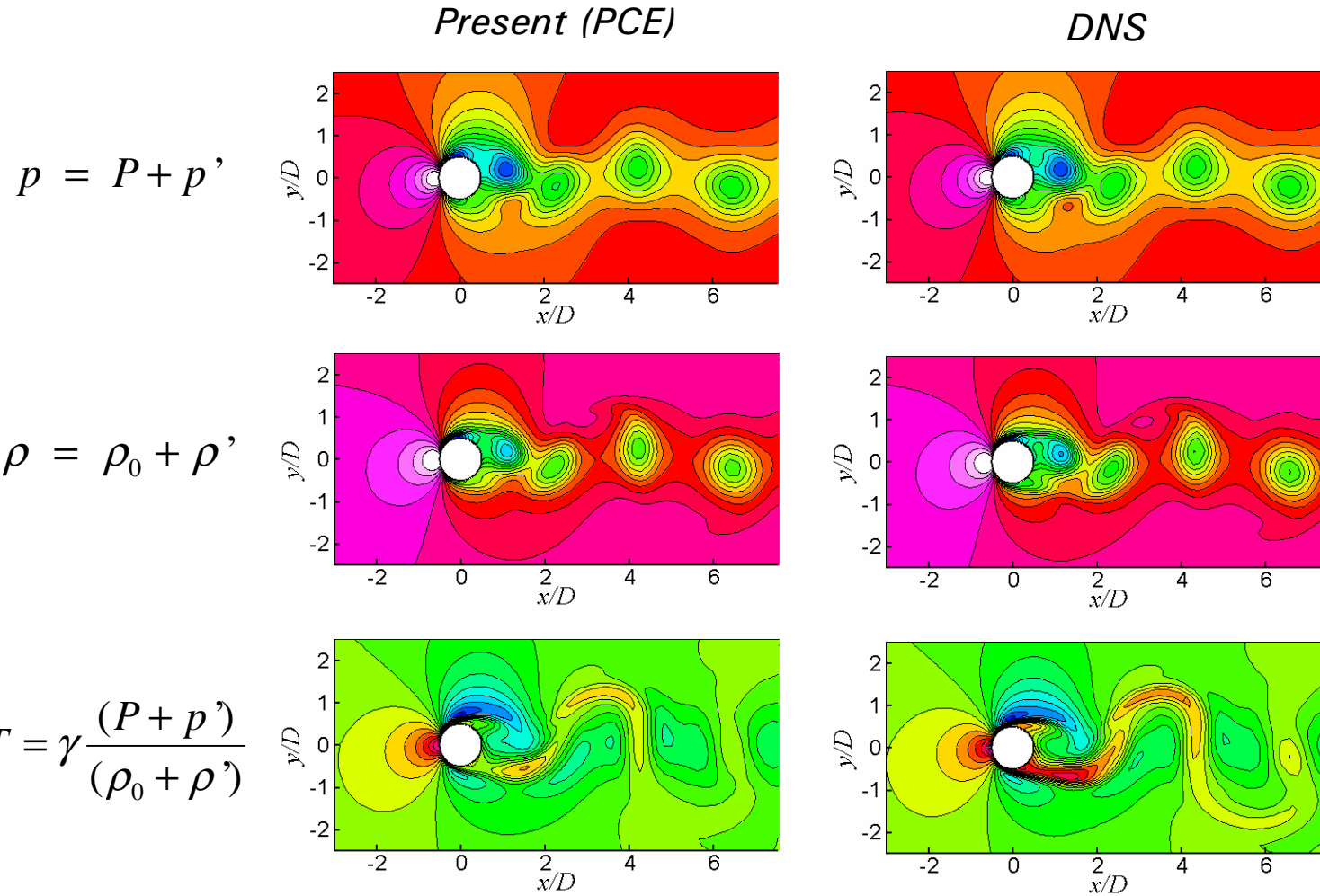
$$w = \Omega + w'$$



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# Validation – near-field compressible flow fields (*re-constructed*)

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## II. Computational Issues of *PCE*



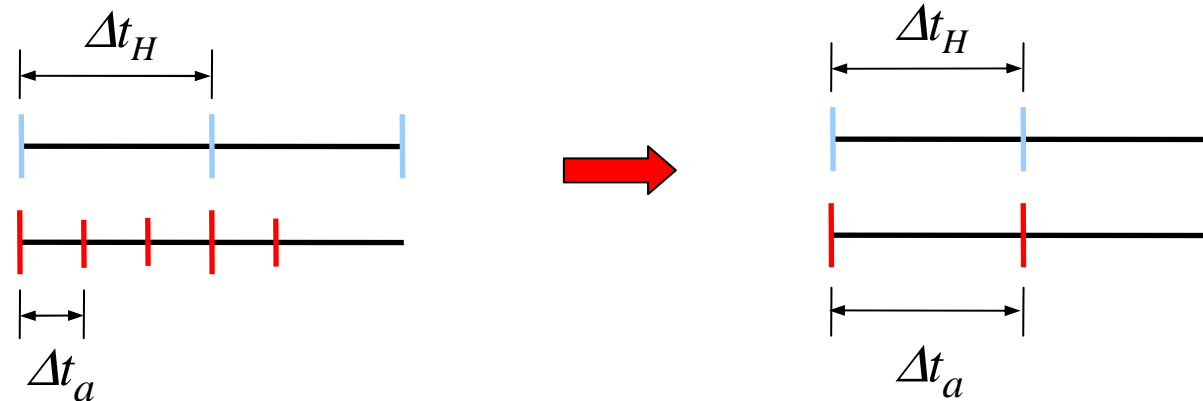


## Computational issues of PCE – efficiency

- As approach to low Mach number limits,
    - severe restriction on **time step** by numerical stability condition.
  - For the turbulent noise prediction,
    - severe restriction on **grid spacing** by flow physics.
- Prefer to use different grid systems for flow and acoustic fields (grid-splitting).

- Increase the minimum normal grid spacing near the wall, in order to allow a larger time step.

$$\Delta x_{\min,A} \square \Delta x_{\min,H} / M_\infty$$

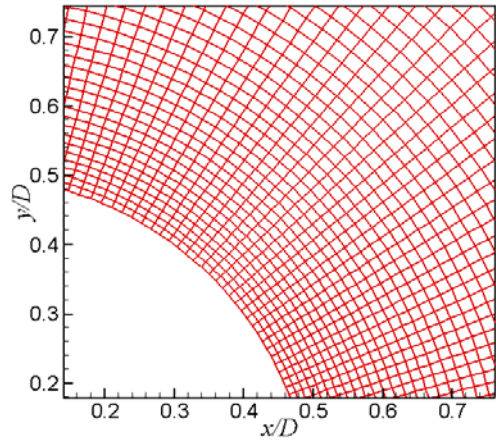


$$\Delta x_{\min,A} = \Delta x_{\min,H}$$

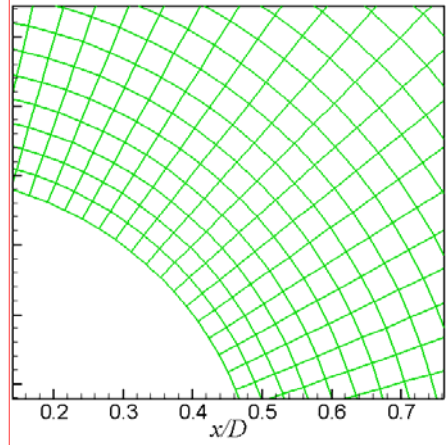
$$\Delta x_{\min,A} \square \Delta x_{\min,H} / M_\infty$$



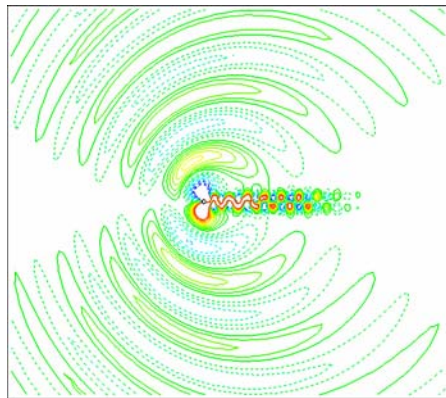
# Cylinder tone (laminar)



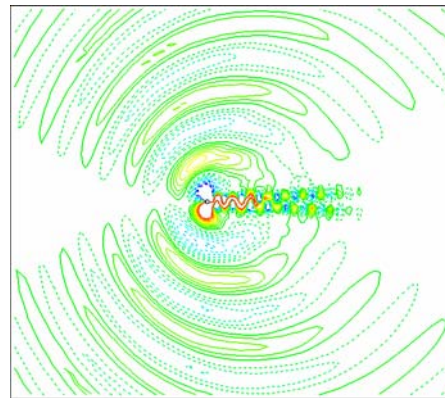
*hydrodynamic*  
(201×201)



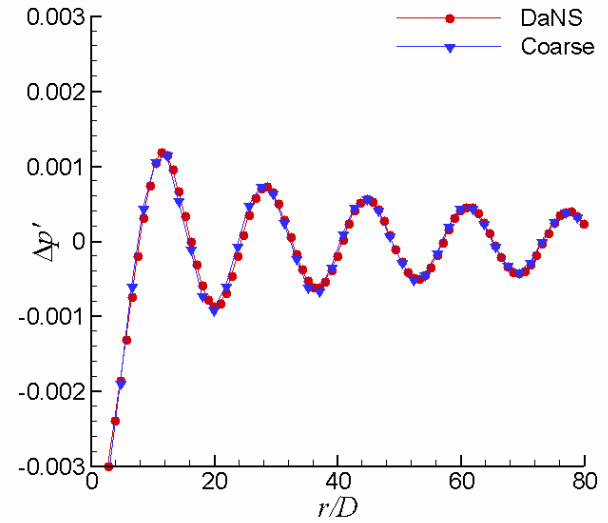
*acoustic (coarse)*  
(121×151)



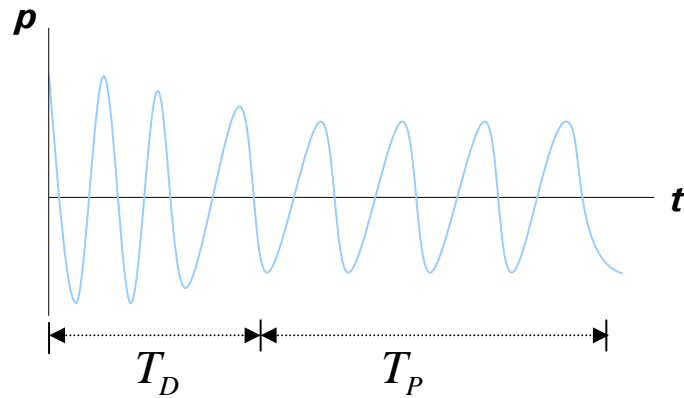
*DNS*



*Present (coarse)*



# Speed-up factor of PCE, $\Psi_{PCE} = T_{DNS} / T_{PCE}$

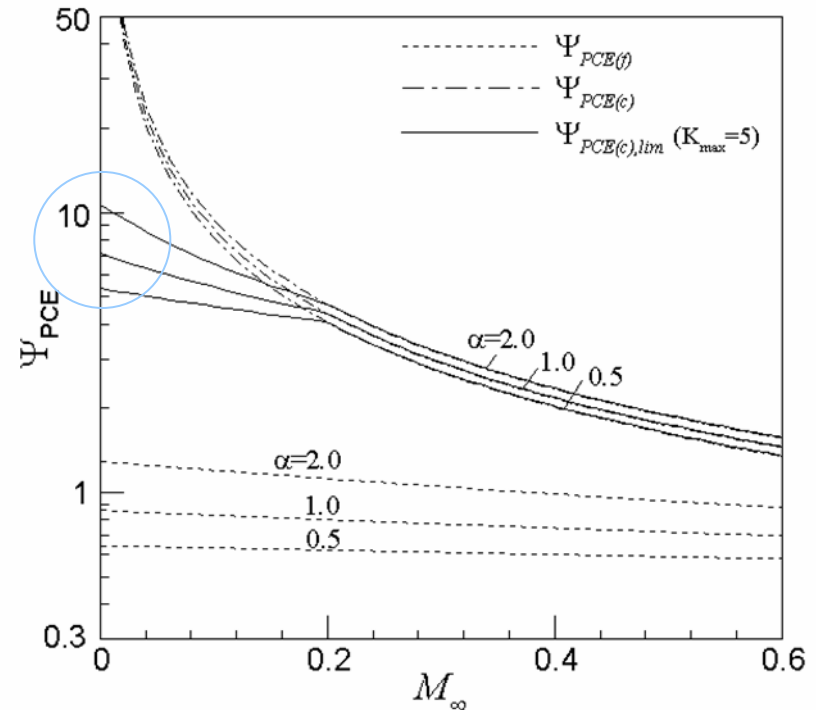


$\alpha : T_D / T_P$  (D: devlp, P: periodic)  
 N : number of periods  
 n : number of iterations per period  
 $t_I$  : CPU time required per iteration  
 (based on a time step for INS)

$$T_{DNS} = \frac{1}{M_\infty} N n t_{I,DNS} + \frac{1}{M_\infty} (\alpha N n t_{I,DNS})$$

$$T_{PCE(f)} = \frac{1}{M_\infty} N n (t_{I,INS} + t_{I,PCE(f)}) + \alpha N n t_{I,INS}$$

$$T_{PCE(c)} = N n (t_{I,INS} + t_{I,PCE(c)}) + \alpha N n t_{I,INS}$$

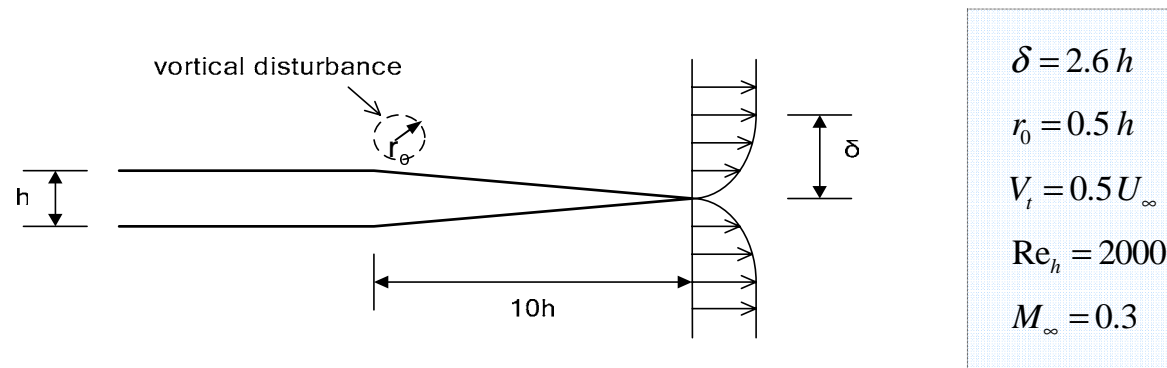


$$\Psi_{PCE(c),lim} = \frac{(\alpha + 1) t_{I,DNS}}{(\alpha M_\infty + 1/k) t_{I,INS} + (1/k) t_{I,PCE(c)}}$$

where  $k = \min(1/M_\infty, K_{max})$ ,  $K_{max}=5$

## Computational issues of PCE – accuracy

- Vortex (or turbulent eddy) scattering from the trailing-edge:
  - transient transformation from quadruple to dipole



### vortical disturbance

$$u_v^* = V_t (y - y_0) \exp\left(\frac{1 - [r/r_0]^2}{2}\right)$$

$$v_v^* = -V_t (x - x_0) \exp\left(\frac{1 - [r/r_0]^2}{2}\right)$$

where

$$r = \sqrt{(x - x_0)^2 + (y - y_0)^2}$$

$$(x_0, y_0) = (-8h, h)$$

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# Acoustic grids

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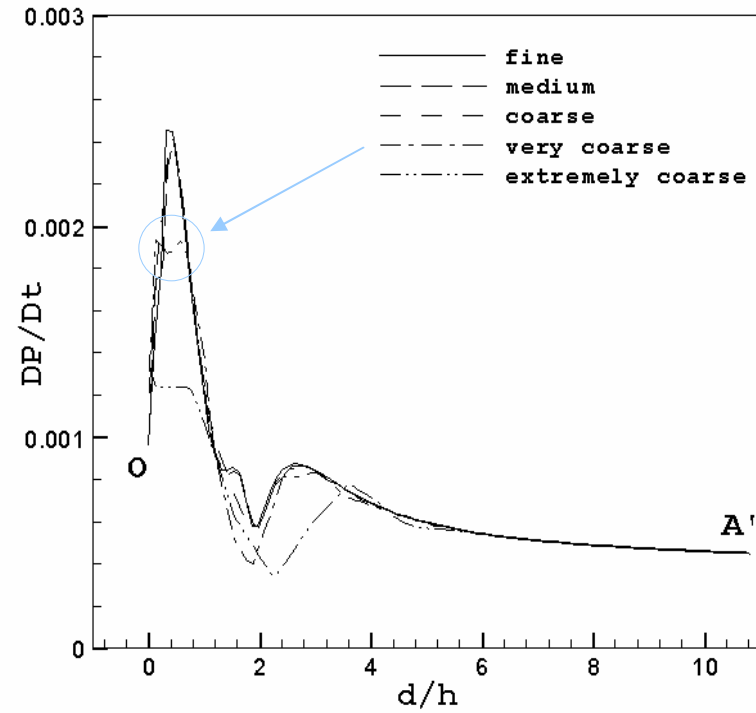
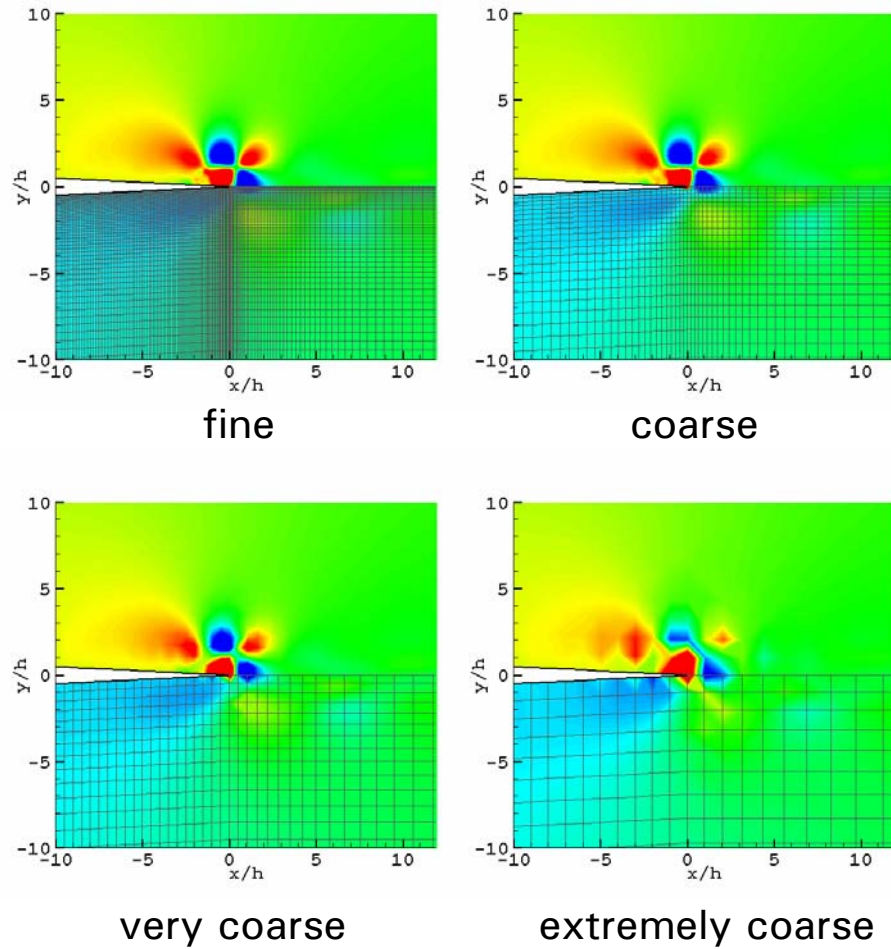
grid coarseness factors

$$R_{\min} = \frac{\Delta x_{\min,A}}{\Delta x_{\min,H}} \quad R_{\delta} = \frac{N_{\delta,H}}{N_{\delta,A}}$$

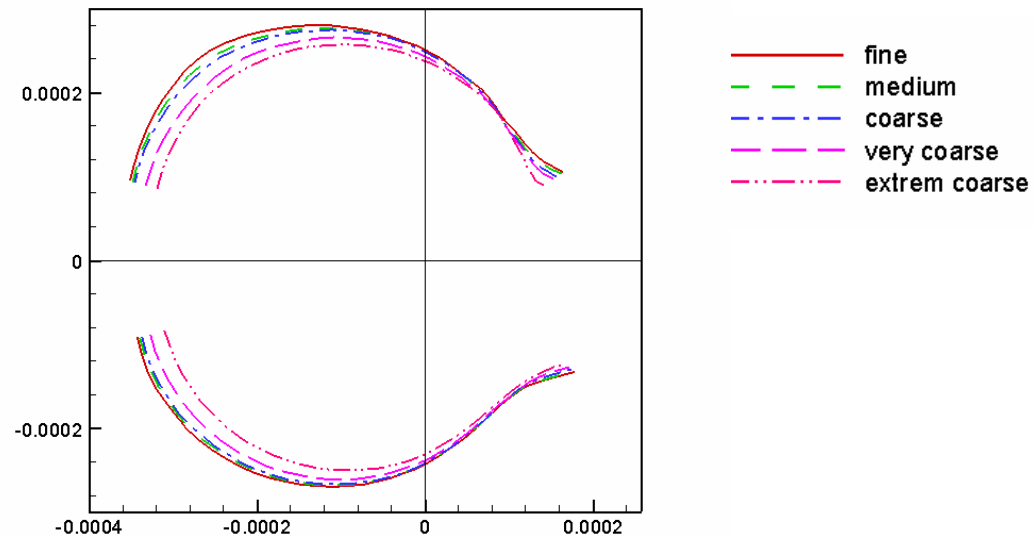
	$N_T$	$N_{\delta}$	$R_{\min}$	$R_{\delta}$
<b>Fine (<i>Hydro</i>)</b>	251×201	26	1	1
<b>Medium</b>	171×161	17	2	1.5
<b>Coarse</b>	116×121	10	4	2.6
<b>Very coarse</b>	101×101	5	10	5.2
<b>Extreme coarse</b>	91×81	3	20	8.6



# $DP/Dt$ field projection



# Acoustic pulse strength and $\Delta p'(rms)$ directivity



	$R_{min}$	$R_{\delta}$	Peak value of $\Delta p'_{rms}$	Relative error	Directivity of $\Delta p'_{rms}$ peak
<b>Fine</b>	1	1	3.67e-4	-	155.1°
<b>Medium</b>	2	1.5	3.63e-4	1.1%	156.4°
<b>Coarse</b>	4	2.6	3.60e-4	1.9%	157.3°
<b>Very coarse</b>	10	5.2	3.46e-4	5.7%	160.2°
<b>Extreme coarse</b>	20	8.6	3.32e-4	9.5%	162.6°



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**III. Linearized Perturbed Compressible Equations (*LPCE*) –  $M \lll 1$**

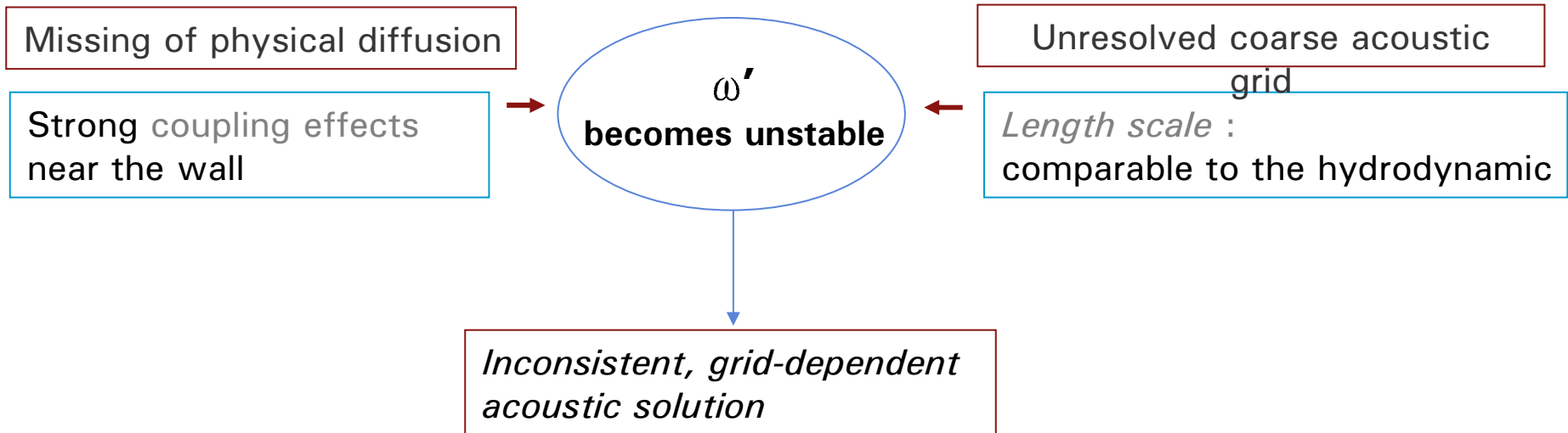




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## Instability of perturbed vorticity ( $\omega'$ )

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*At low Mach numbers,*  
the effect of perturbed vorticity is negligible for the noise generation.

*Exclude the errors caused by perturbed vorticity  
by suppressing the generation of perturbed vorticity !*



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## Perturbed Compressible Eqs. (PCE)

---

- $\frac{\partial \rho'}{\partial t} + \frac{\partial}{\partial x_i}(\rho u'_i + \rho' U_i) = 0$
- $\frac{\partial}{\partial t}(\rho u'_i + \rho' U_i) + \frac{\partial}{\partial x_j} [u_i(\rho u'_i + \rho' U_i) + \rho_0 U_i u'_j + p' \delta_{ij}] = \frac{\partial \tau'_{ij}}{\partial x_j}$ ,
- $\frac{\partial p'}{\partial t} + u_j \frac{\partial p'}{\partial x_j} + \gamma p \frac{\partial u'_j}{\partial x_j} = -\frac{DP}{Dt} + (\gamma - 1) \left( \phi - \frac{\partial q_j}{\partial x_j} \right)$

where  $\tau'_{ij} = \mu_0 \left( \partial_j u'_i + \partial_i u'_j - \frac{2}{3} \delta_{ij} \partial_k u'_k \right)$

- No-slip condition at the wall



---

## Derivation of LPCE formulation

---

- Neglecting the second-order, non-linear terms

$$\frac{\partial \rho'}{\partial t} + (\vec{U} \cdot \nabla) \rho' + \rho_0 (\nabla \cdot \vec{u}') = 0$$

$$\frac{\partial \vec{u}'}{\partial t} + \nabla(\vec{u}' \cdot \vec{U}) + \frac{1}{\rho_0} \nabla p' = -(\vec{\Omega} \times \vec{u}') - (\vec{\omega} \times \vec{U}) - \frac{\rho'}{\rho_0} \frac{D\vec{U}}{Dt} + \frac{1}{\rho_0} \vec{f}'_{vis}$$

*Mathematical identity*

$$(\vec{U} \cdot \nabla) \vec{u}' + (\vec{u}' \cdot \nabla) \vec{U} = \nabla(\vec{u}' \cdot \vec{U}) + (\vec{\Omega} \times \vec{u}') + (\vec{\omega} \times \vec{U})$$

$$\frac{\partial p'}{\partial t} + (\vec{U} \cdot \nabla) p' + \gamma P (\nabla \cdot \vec{u}') + (\vec{u}' \cdot \nabla) P = -\frac{DP}{Dt} + (\gamma - 1)(\Phi - \nabla \cdot \vec{q})$$



## Perturbed vorticity transport eq. (3D)

$$\frac{\partial}{\partial t} \left( \frac{\vec{\omega}'}{\rho} \right) + (\vec{u}' \cdot \nabla) \left( \frac{\vec{\omega}'}{\rho} \right) = \frac{1}{\rho} [(\vec{\Omega} \cdot \nabla) \vec{u}' + (\vec{\omega}' \cdot \nabla) \vec{u}] - \frac{1}{\rho} [(\vec{u}' \cdot \nabla) \vec{\Omega} + \vec{\Omega}(\nabla \cdot \vec{u}')] + \frac{1}{\rho^3} (\nabla \rho \times \nabla p) + \frac{1}{\rho} \nabla \times \vec{F}'_{vis}$$

$$\frac{\partial \vec{u}'}{\partial t} + \nabla(\vec{u}' \cdot \vec{U}) + \frac{1}{\rho_0} \nabla p' = \underbrace{- (\vec{\Omega} \times \vec{u}') - (\vec{\omega}' \times \vec{U})}_{\text{Sources to the perturbed vorticity generation}} - \frac{\rho'}{\rho_0} \frac{D\vec{U}}{Dt} + \frac{1}{\rho_0} \vec{f}'_{vis}$$

Sources to the perturbed vorticity generation



---

# Linearized *Perturbed Compressible Eqs. (LPCE)* - $M \ll 1$

---

- Linearized to the base incompressible flow field

$$\frac{\partial \rho'}{\partial t} + (\vec{U} \cdot \nabla) \rho' + \rho_0 (\nabla \cdot \vec{u}') = 0$$

$$\frac{\partial \vec{u}'}{\partial t} + \nabla(\vec{u}' \cdot \vec{U}) + \frac{1}{\rho_0} \nabla p' = 0 \quad \dots \rightarrow \boxed{\frac{\partial \omega'}{\partial t} = 0}$$

$$\frac{\partial p'}{\partial t} + (\vec{U} \cdot \nabla) p' + \gamma P (\nabla \cdot \vec{u}') + (\vec{u}' \cdot \nabla) P = \boxed{-\frac{DP}{Dt}}$$

Only explicit noise source at low Mach numbers  
*Goldstein(2003), Ewert & Schröder (2003)*



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# Numerical test problems

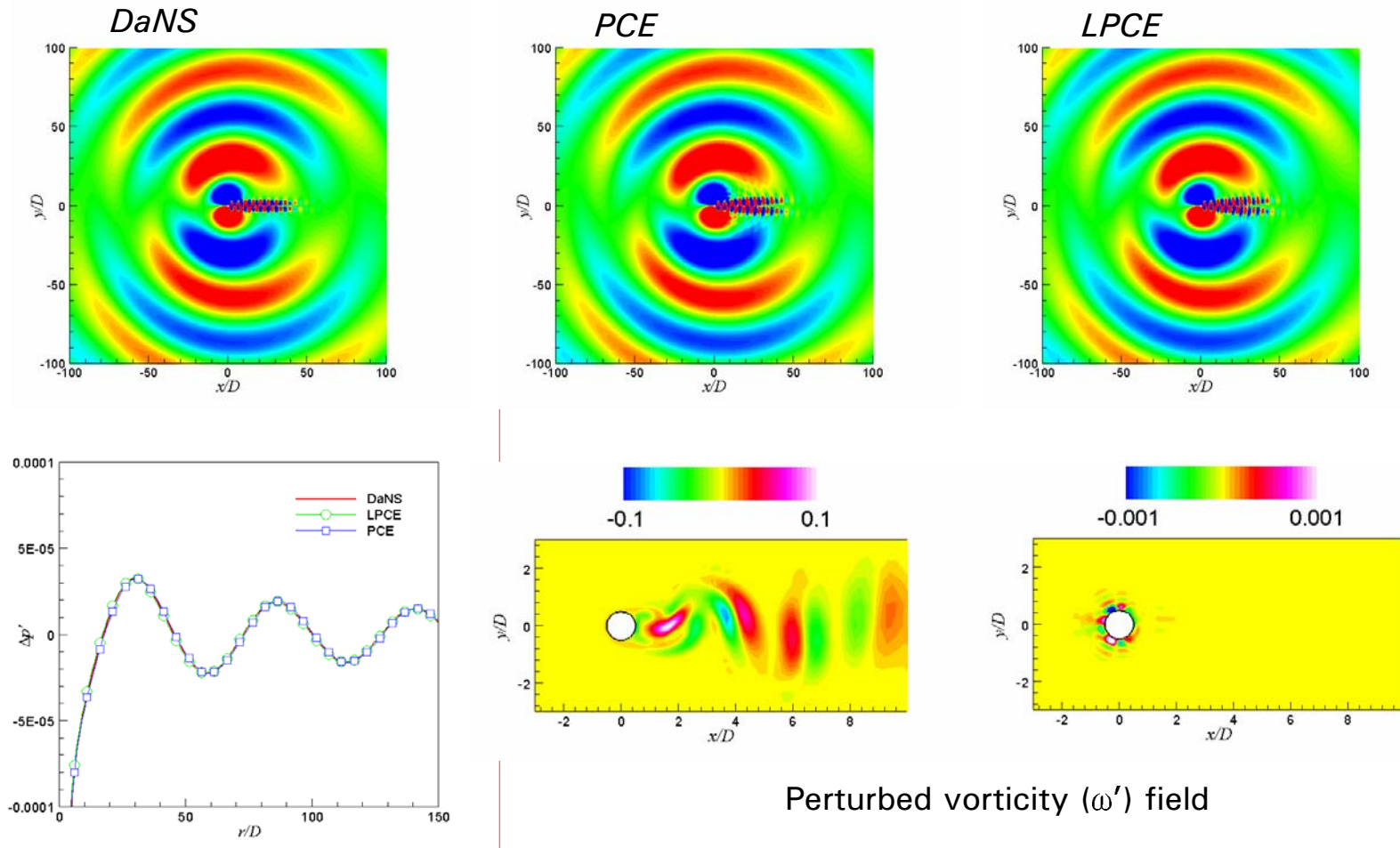
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1. Dipole tone noise from a circular cylinder at  $Re = 150$ ,  $M = 0.1$ 
  - Basic validation by comparing with ***DNS***
2. Quadruple noise generated by the Kirchhoff vortex
  - Fundamental vortex sound problem, comparing with the ***exact solution***
3. Sound radiated from a temporal mixing layer
  - Model for ***turbulent noise*** prediction, practical vortex sound problem

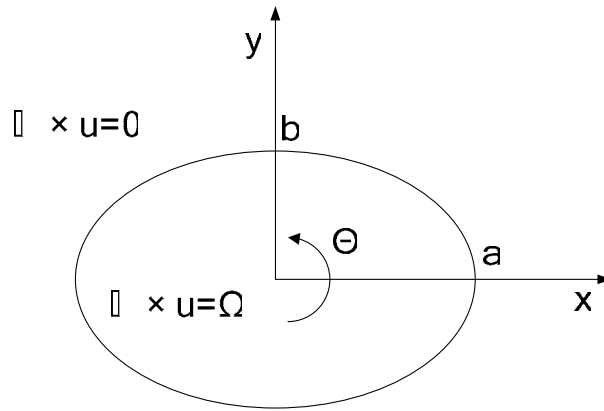


# 1. Tonal noise from a circular cylinder

- Tonal noise from a circular cylinder at  $M_\infty = 0.1$ ,  $Re_D = 150$  ( $\Delta p'$ )



## 2. Quadruple noise generated by the Kirchhoff vortex



Kirchhoff vortex

$$\Theta = \frac{ab}{(a+b)^2} \Omega$$

$$M_{\Theta} = \frac{(a+b)\Theta}{c} = 0.1$$

*Acoustic grid resolution*

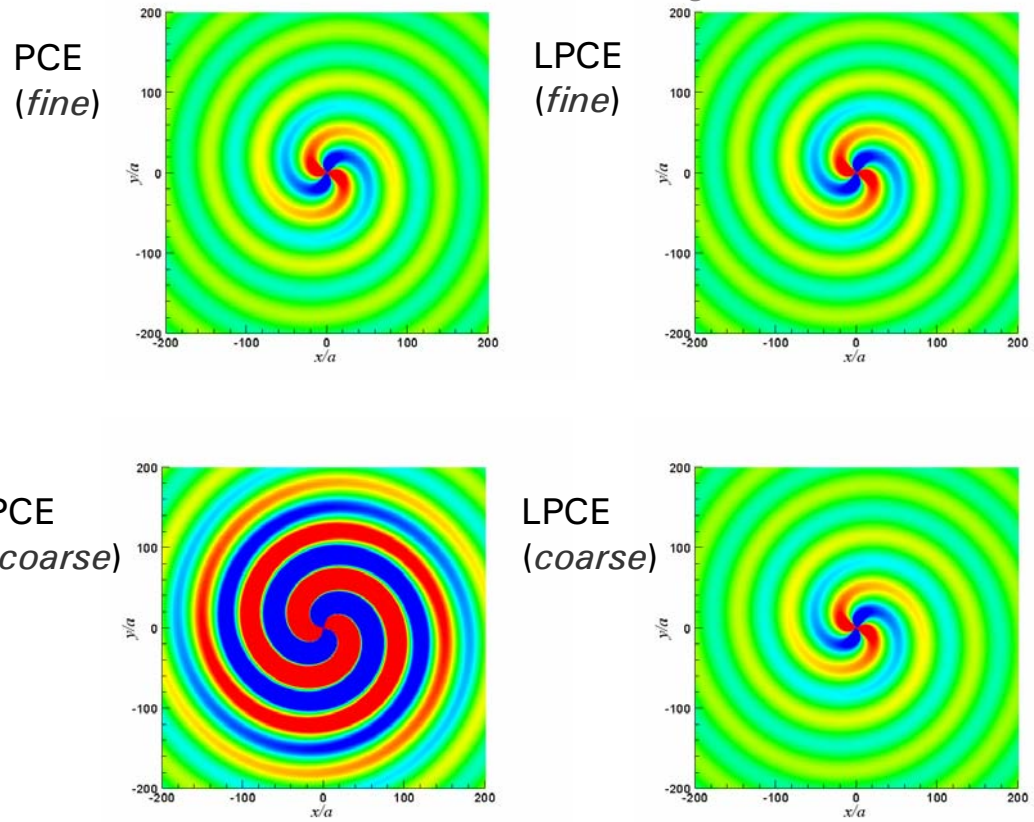
	Fine	Coarse
Kirchhoff vortex core	100×100	10×10



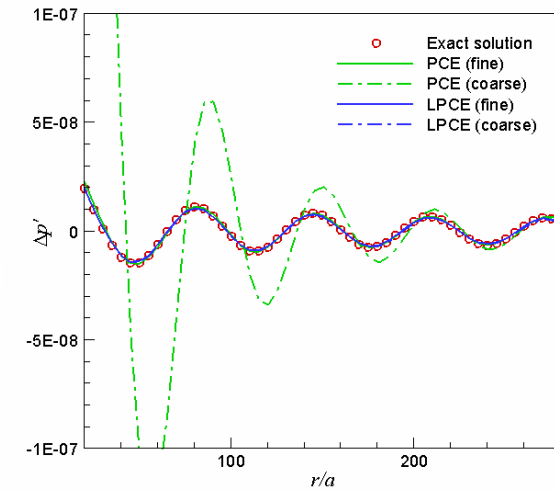


## 2. Quadruple noise generated by the Kirchhoff vortex

*no perturbed vorticity generation*



Instantaneous pressure fluctuation along the  $y=0$  line



*Exact solution by Müller (1998)*

*Acoustic solution is corrupted by falsely resolved perturbed vorticity!*

Pressure fluctuation ( $\Delta p'$ ) fields

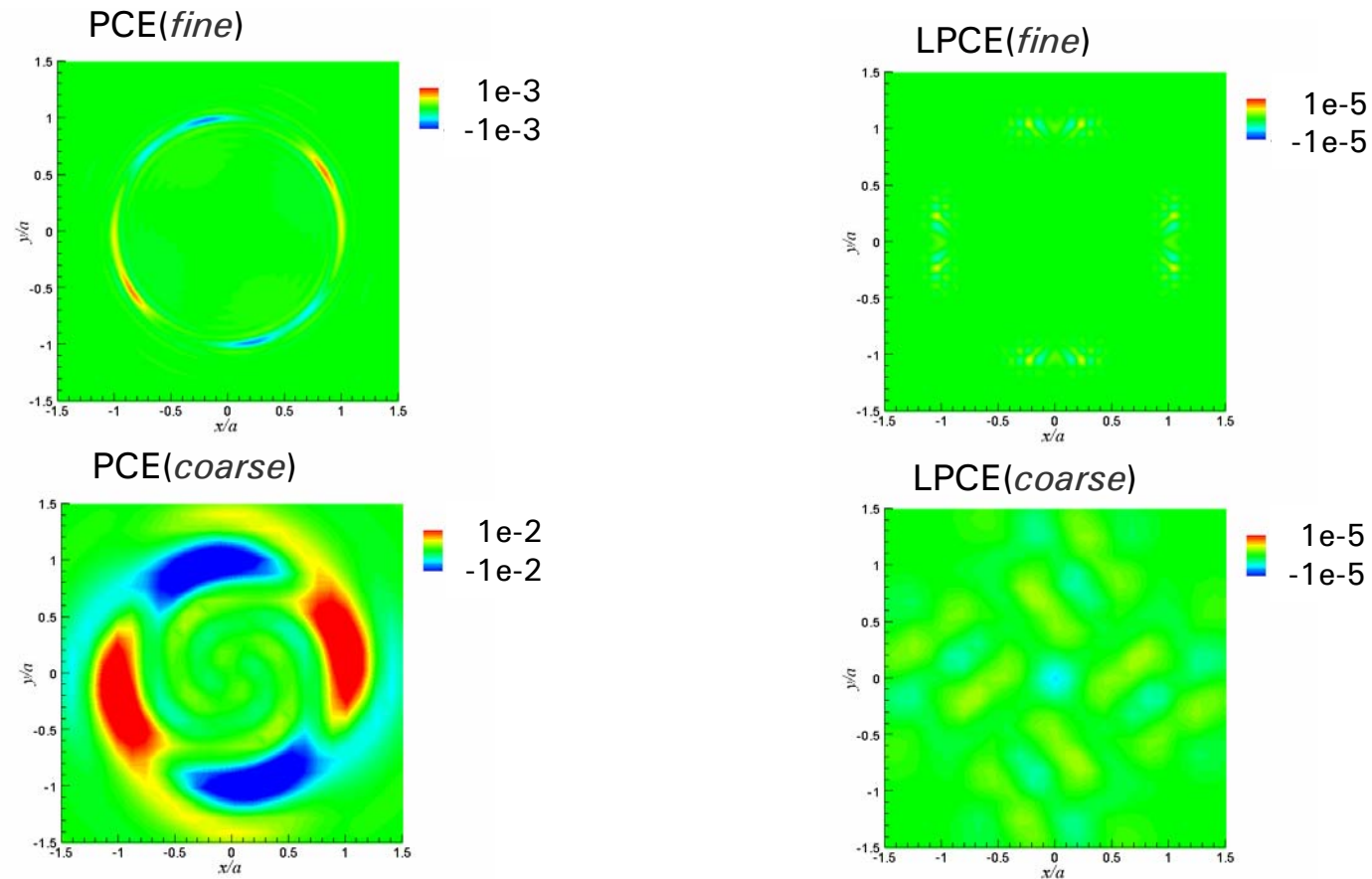


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## 2. Quadruple noise generated by the Kirchhoff vortex

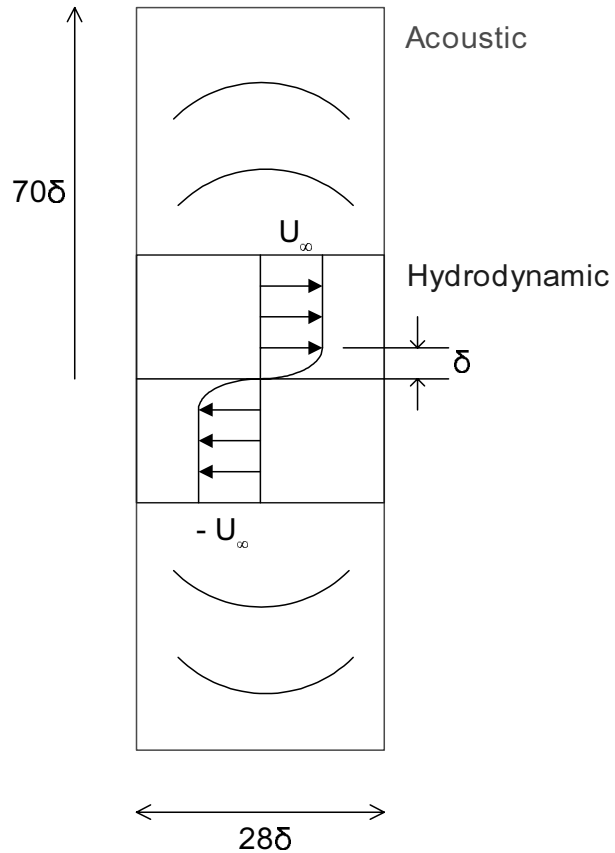
---

Most dominant source for the perturbed vorticity ( $\omega'$ ) generation :  $(\vec{u}' \cdot \nabla)\Omega$



Perturbed vorticity ( $\omega'$ ) fields

### 3. Sound radiated from a temporal mixing layer



$$M_s = \frac{2U_\infty}{c} = 0.1$$

$$Re_\delta = 10000$$

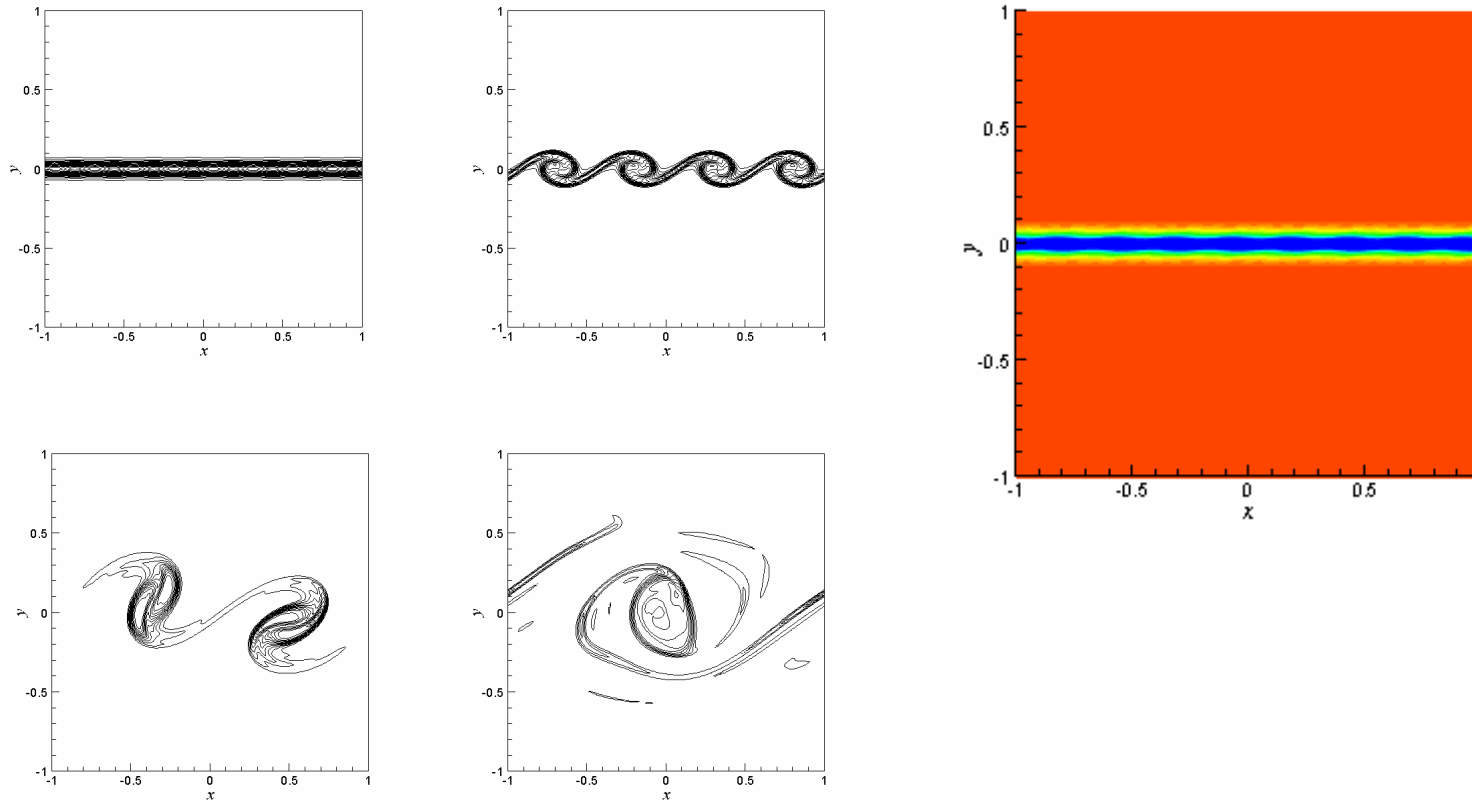
*Acoustic grid resolution*

	Fine	Coarse
Hydrodynamic domain	200×200	40×40



### 3. Sound radiated from a temporal mixing layer

- The evolution of vortical field ( $\Omega$ )

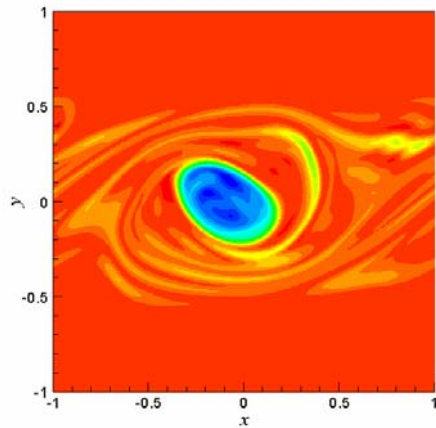


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### 3. Sound radiated from a temporal mixing layer

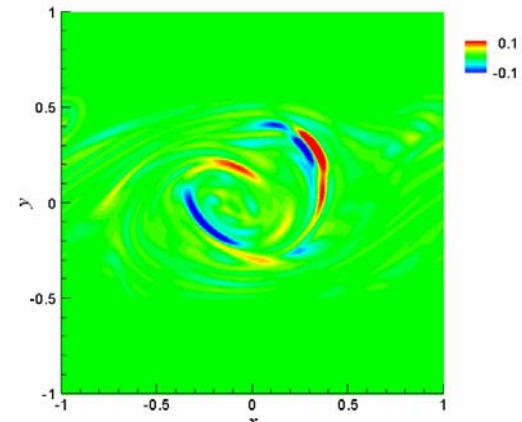
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Hydrodynamic vorticity ( $\Omega$ )

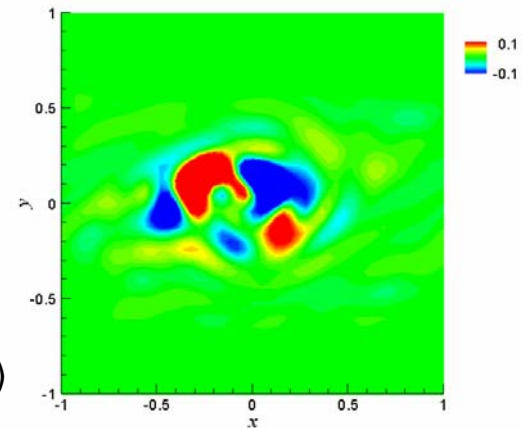


Perturbed vorticity ( $\omega'$ )

PCE (*fine*)



PCE (*coarse*)

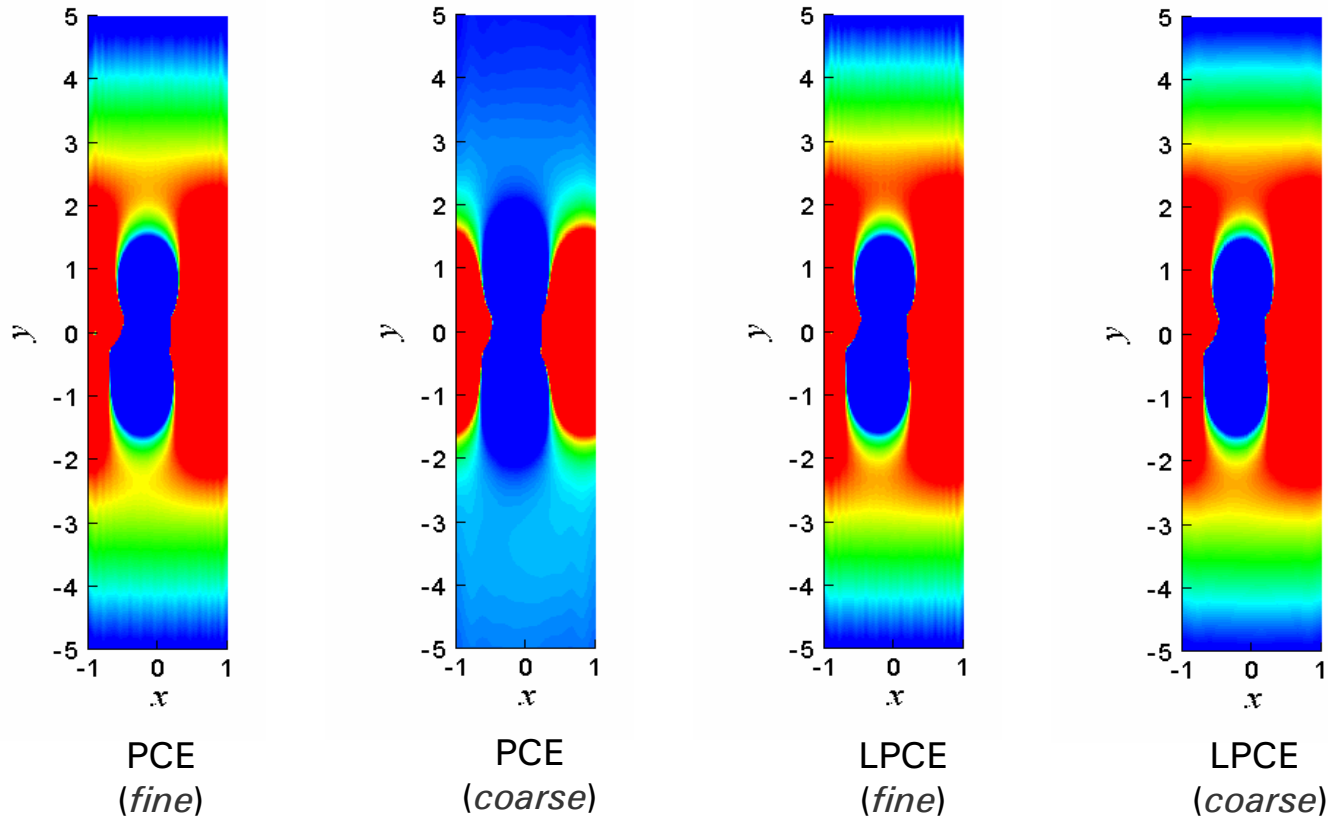


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### 3. Sound radiated from a temporal mixing layer

---

- Acoustic fields ( $\Delta p'$ )

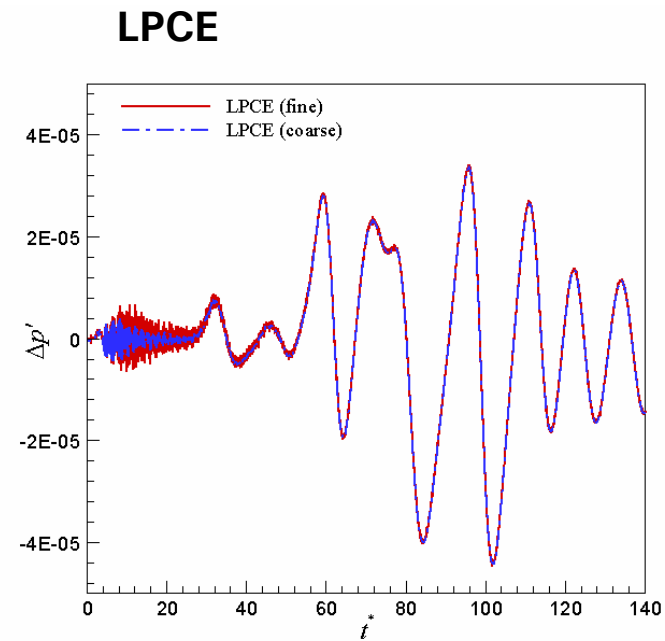
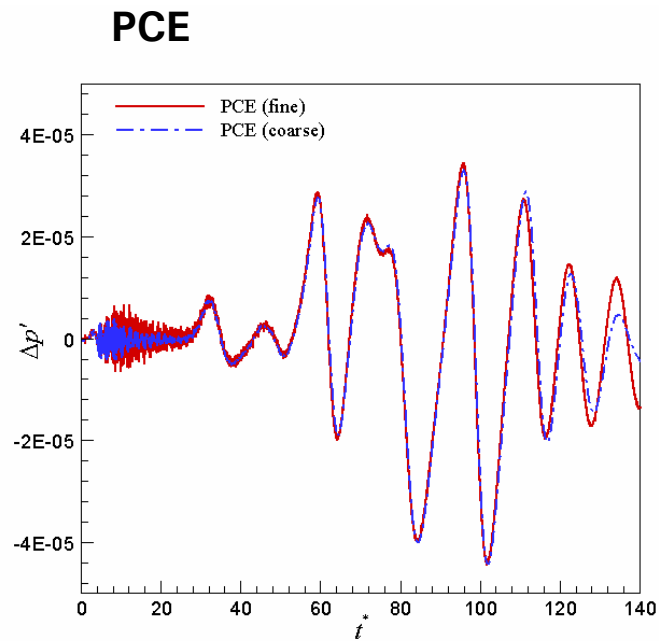


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### 3. Sound radiated from a temporal mixing layer

---

- Time histories of instantaneous pressure fluctuation ( $\Delta p'$ ) at  $(0, 56\delta)$



---

## Concluding Remarks

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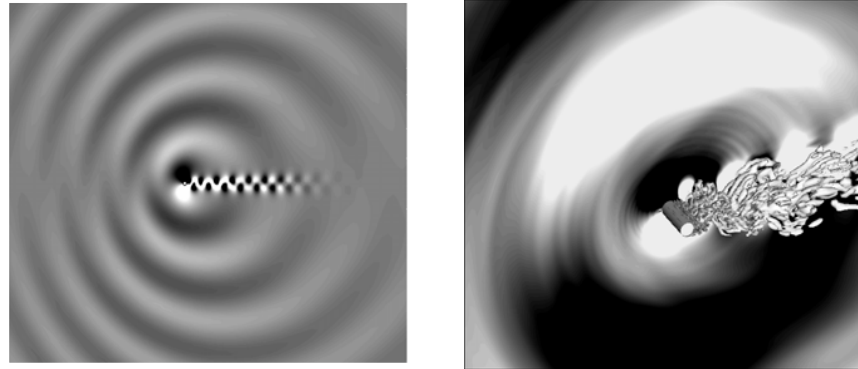
- The perturbed vorticity is not a *major* contributor in acoustic wave generation at low Mach numbers.
- The perturbed vorticity could be a source of unwanted errors on the *unresolved grid*.
- The '*Linearized Perturbed Compressible Equations*' in which the generation of perturbed vorticity is *suppressed*, are proposed.
- The consistent, grid-independent acoustic solution is secured by the *LPCE formulation*.
- The formulation may be useful for the *turbulent noise prediction at low Mach numbers*.





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**Computational Aeroacoustics**  
*for*  
**Low Mach Number Flows (II)**



**Young J. Moon**

Korea University  
Department of Mechanical Engineering  
Seoul, Korea

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# Contents

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- I. Perturbed Compressible Equations (*PCE*) –  $M \ll 1$
- II. Computational Issues of *PCE* – *efficiency & accuracy*
- III. Linearized Perturbed Compressible Equations (*LPCE*) –  $M \ll \ll 1$
- IV. 3D Flow past a Circular Cylinder –  $Re_D = 46000$  &  $M = 0.21$
- V. Phonation Aero-acoustics in Vocal Tract
- VI. Concluding Remarks



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**III. 3D Flow past a Circular Cylinder at  $Re_D = 46000$  &  $M = 0.21$**



---

## 3D Flow past a circular cylinder at $Re_D = 46000$ & $M = 0.21$

---

- $Re_D = 46000$  : sub-critical regime ( $1,000 < Re_D < 200,000$ ).
- A boundary layer remains laminar but the shear layer exhibits immediate transition to turbulence.
- It produces a broadened tone at vortex shedding frequency as well as broadband noise due to wake turbulences.
- Spanwise correlation length is about  $2.7D$  (Szepessy, 1994).



---

# Incompressible LES

---

- Filtered incompressible Navier-Stokes equations.

$$\frac{\partial \tilde{U}_j}{\partial x_j} = 0$$
$$\rho_0 \frac{\partial \tilde{U}_i}{\partial t} + \rho_0 \frac{\partial}{\partial x_j} (\tilde{U}_i \tilde{U}_j) = -\frac{\partial \tilde{P}}{\partial x_i} + \mu_0 \frac{\partial}{\partial x_j} \left( \frac{\partial \tilde{U}_i}{\partial x_j} + \frac{\partial \tilde{U}_j}{\partial x_i} \right) - \rho_0 \frac{\partial}{\partial x_j} M_{ij}$$

- Smagorinsky sub-grid model with Van Driest wall damping function:  
*Smagorinsky constant*  **$C_s = 0.065$**

$$M_{ij} = \overline{U_i U_j} - \tilde{U}_i \tilde{U}_j$$
$$= -\{ (0.065)(1 - \exp(y^+ / 25)^3)^{1/2} \Delta \}^2 |\tilde{S}| \tilde{S}_{ij}$$

- No model **New Calc.**



---

## Computational methods for *iLES*

---

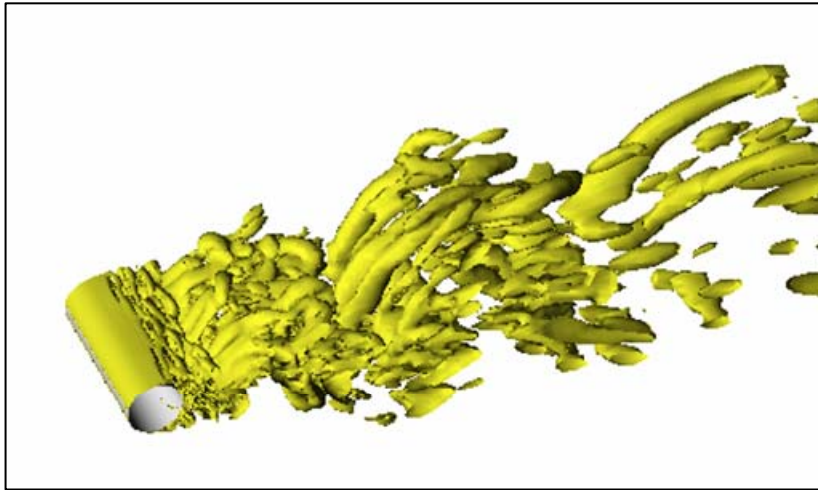
- Iterative fractional step method.
- A sixth-order compact finite difference scheme for spatial discretization.
- 3-stage Runge-Kutta method for time integration.
- Spanwise domain size,  $L_s = 3D$ .
- Uniform inflow condition / Periodic spanwise boundary condition.
- O-grid with  $181 \times 181 \times 31 (= 0.97 \times 10^6)$  points.



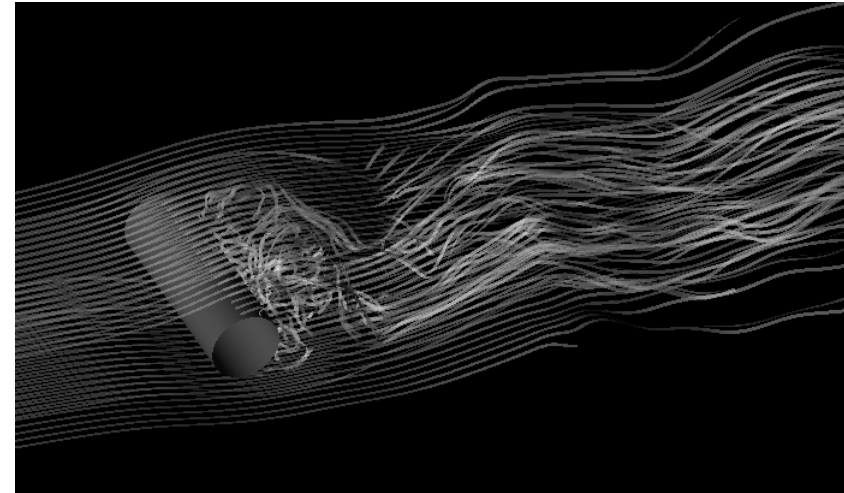
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# Turbulent flow field

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*Iso-surfaces of  $Q = 0.5$*

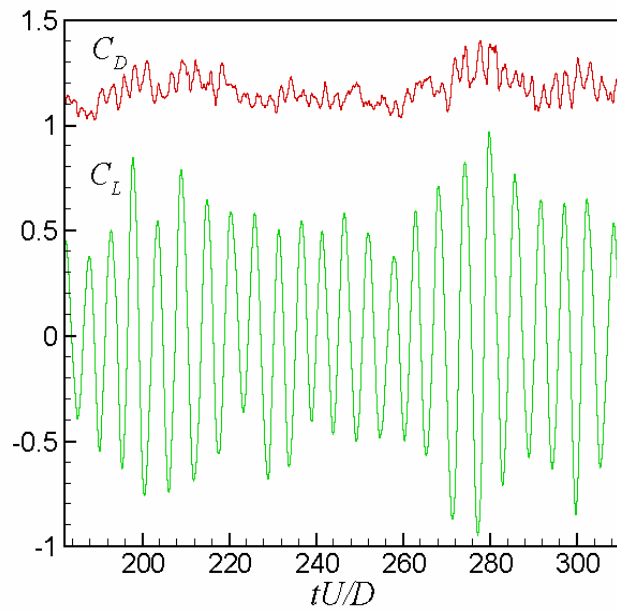


*Streamtraces*

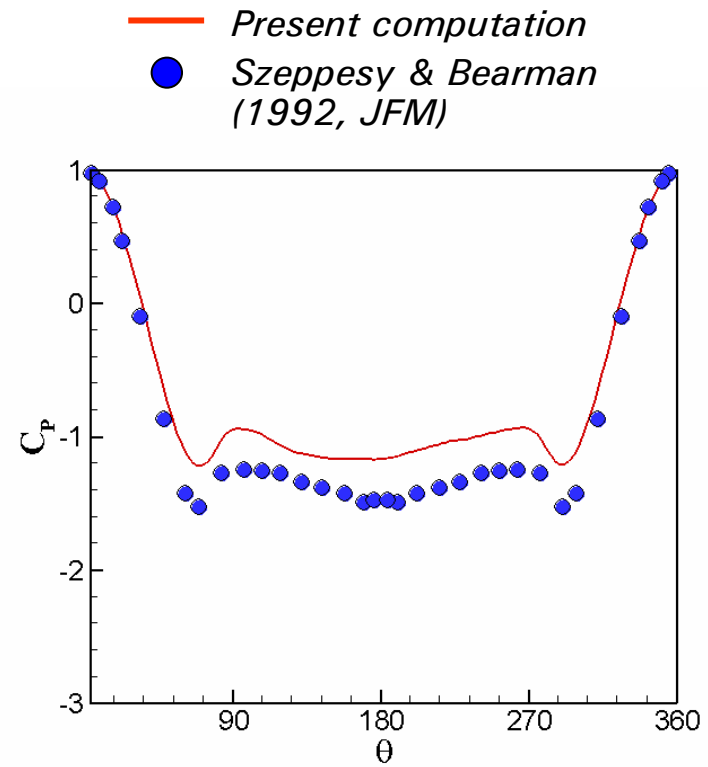
*Highly three-dimensional wake structure and coherent Karman vortex street.*



# Validation – *iLES* (Smagorinsky)



Time histories of drag and lift coefficients



Mean pressure coefficient at the wall





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## Validation – Mean & Fluctuating data

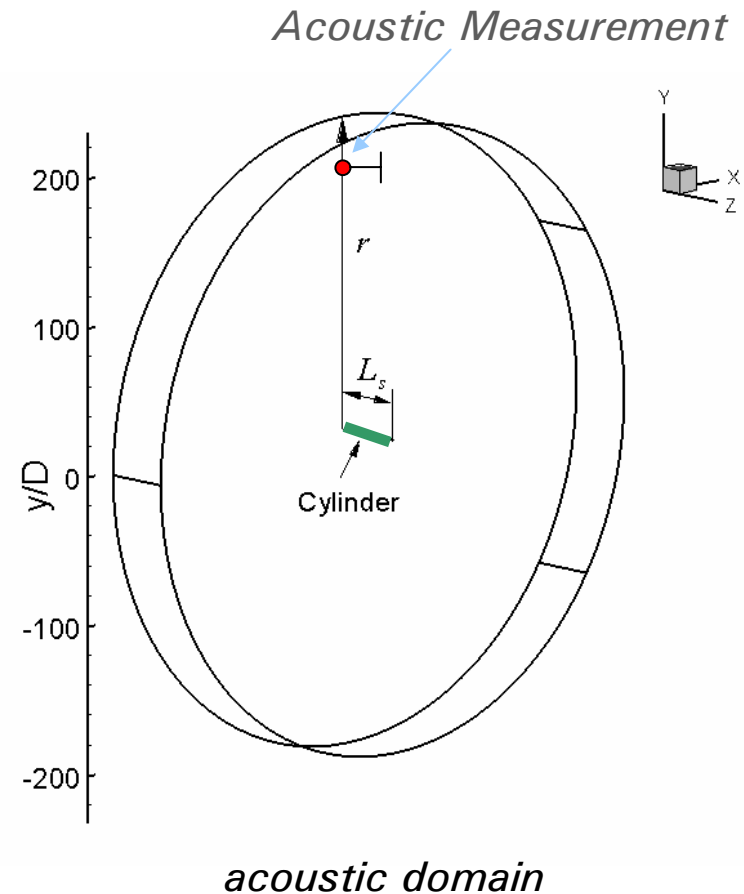
---

	Present LES (Smagorinsky)	Experiment (Szepessy & Bearman)
$St$	0.19	0.19
$C_{D,avg}$	1.15	1.35
$C_{D'rms}$	0.067	0.16
$C_{L'rms}$	0.44	0.45-0.5



## 3D PCE

- Acoustic measurement( $r = 185D$ ):  
*Boudet et al. (AIAA 2003-3217)*.
- Acoustic domain:  
extended to  $r = 200D$ .
- Spanwise domain:  
same as *LES* ( $L_s = 3D$ ).
- Wavelength( $St \sim 0.19$ ):  
about  $25D$ .
- Very narrow domain:  
 $L_s/\lambda \approx 0.12$  and  $r/L_s \approx 66$ .

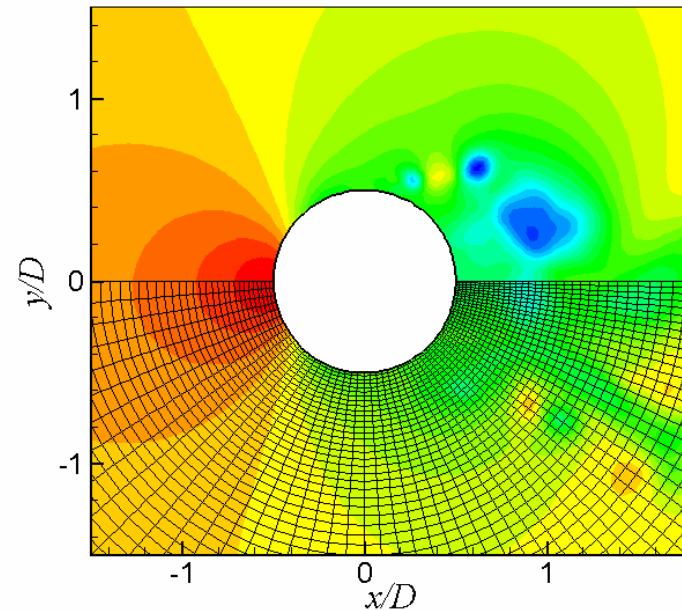


---

## Acoustic grid

---

- $121 \times 181 \times 11 (= 0.24 \times 10^6)$  points.
- 1.5 times coarser in *r-dir*.
- minimal grid spacing at the wall:  
5 times larger than the hydro-grid.
- acoustic calculation:  
with same time step used for  
incompressible LES,  
5 times larger than DaNS.



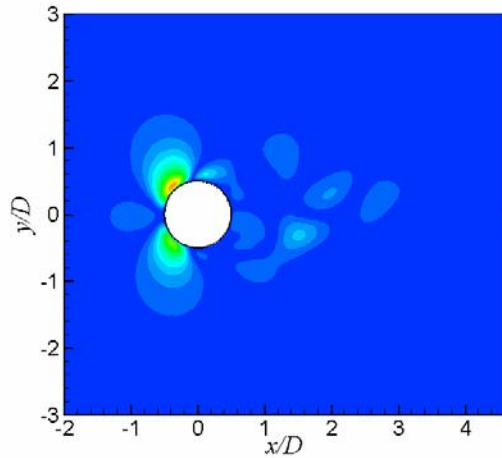
acoustic grid &  
interpolated static pressure field

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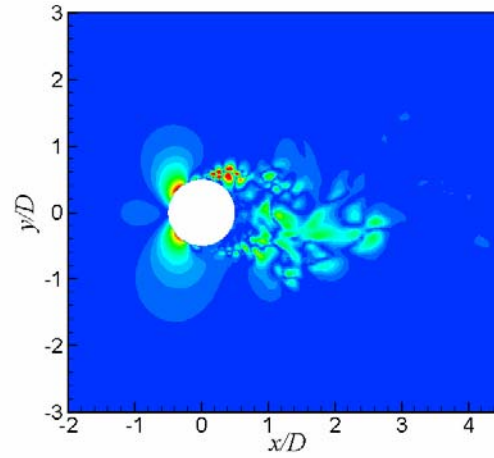
# Explicit source term

---

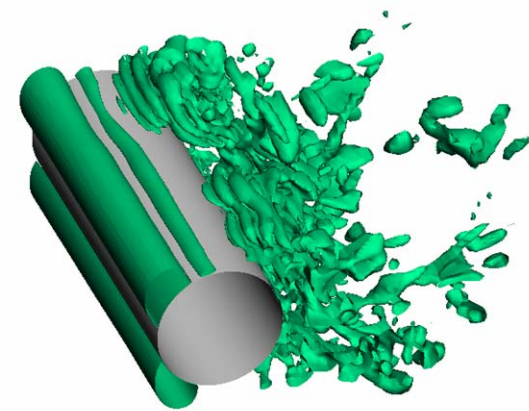
- magnitude of the explicit source term,  $|D\tilde{P}/Dt|$



$Re_D = 200$  (INS)



$Re_D = 46000$  (LES)



---

## Periodic BC v.s. Absorbing BC

---

- Spanwise boundary condition for 3D acoustic simulation of cylindrical problems.
  - Periodic boundary condition
    - Unphysically correlated acoustic solution for a small span.
  - Absorbing (non-reflecting) boundary condition
    - Simulating 3D noise radiation from a small span.
    - Does an absorbing boundary condition, however, work properly for narrow domains ( e.g.  $r / L_s \gg 1$  ) ?



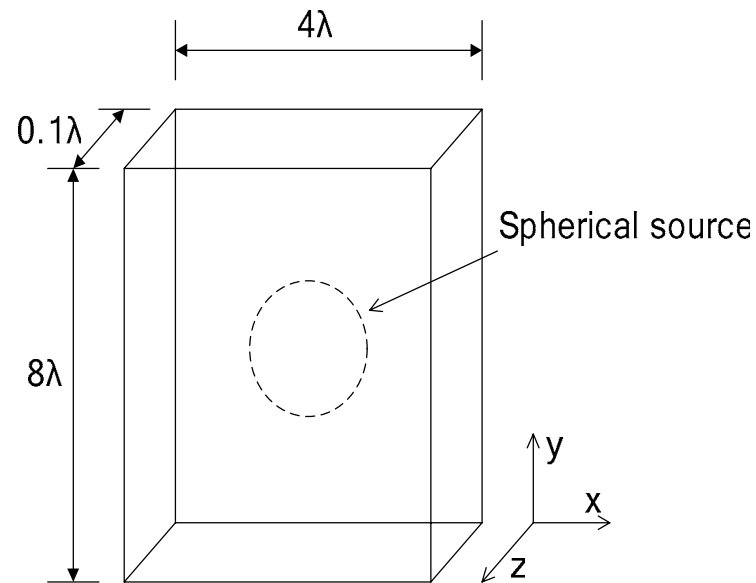
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## 3D Acoustic Wave Propagation in a Narrow Domain

---

$$U = V = W = 0$$

$$P(\bar{x}, t) = P_\infty + 0.1 \sin(2\pi\lambda t) \exp\left(-\frac{x^2 + y^2 + z^2}{25}\right)$$

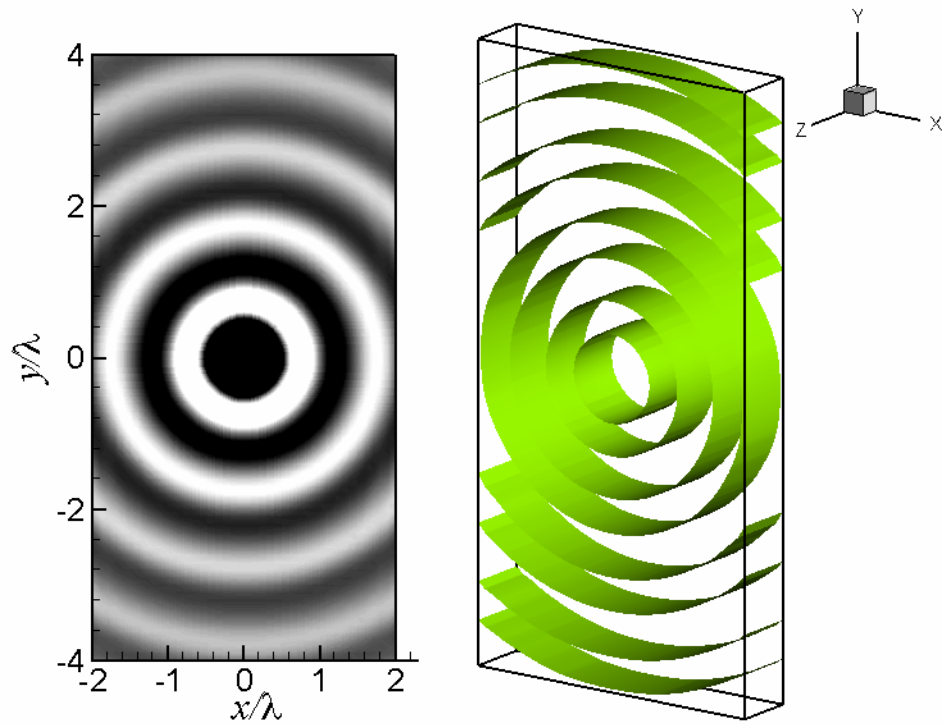


***schematic of numerical exp.***

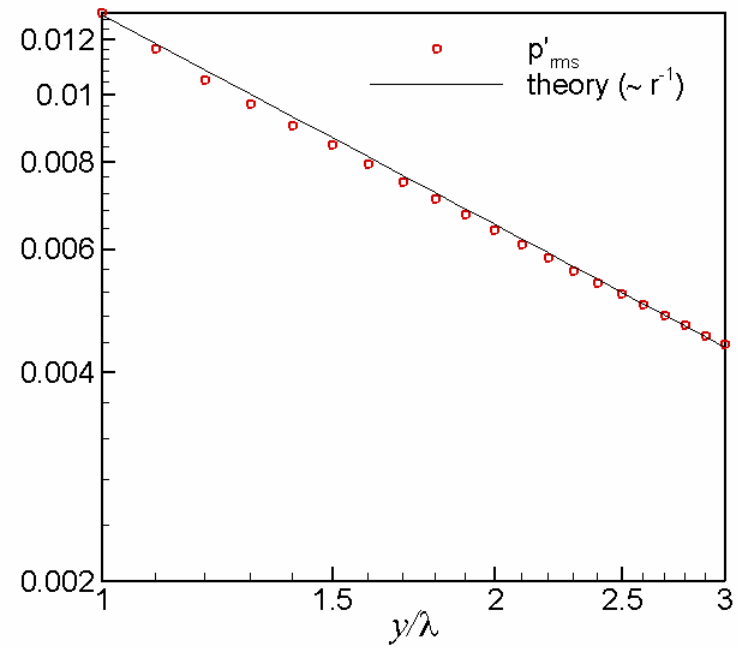
To verify the ETA BC, a numerical experiment is set up.

- A spherical harmonic acoustic source .
- Aspect ratio is about  $r/L_s = 80$  in *yz-plane*.
- Acoustic grid:  $41 \times 81 \times 11$  points.
- PCE is solved with a zero convectational velocity.

# Computational Results



*Contours & Iso-surfaces of pressure fluctuation;  $z$ -axis is magnified by 8 times*

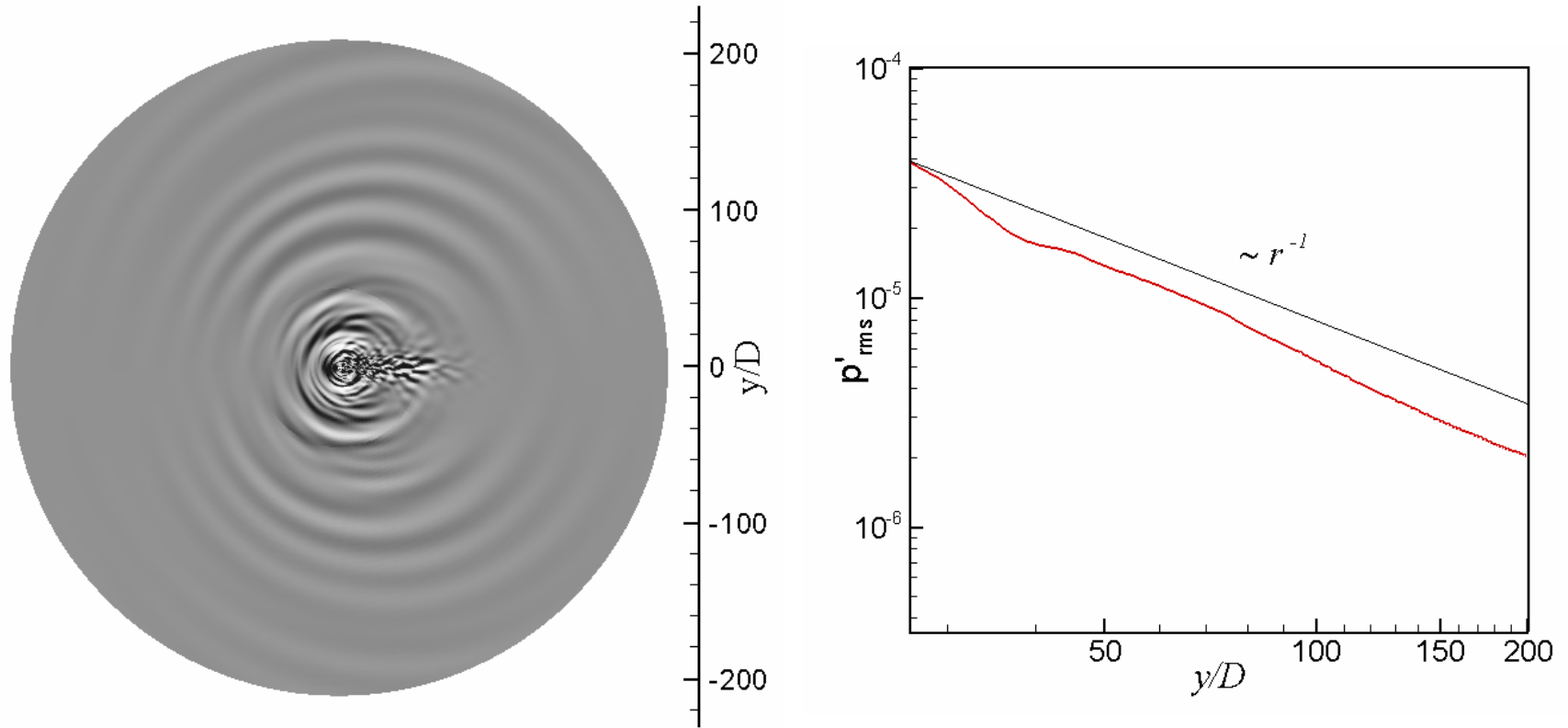


*Acoustic wave strength along the  $y$ -axis*

---

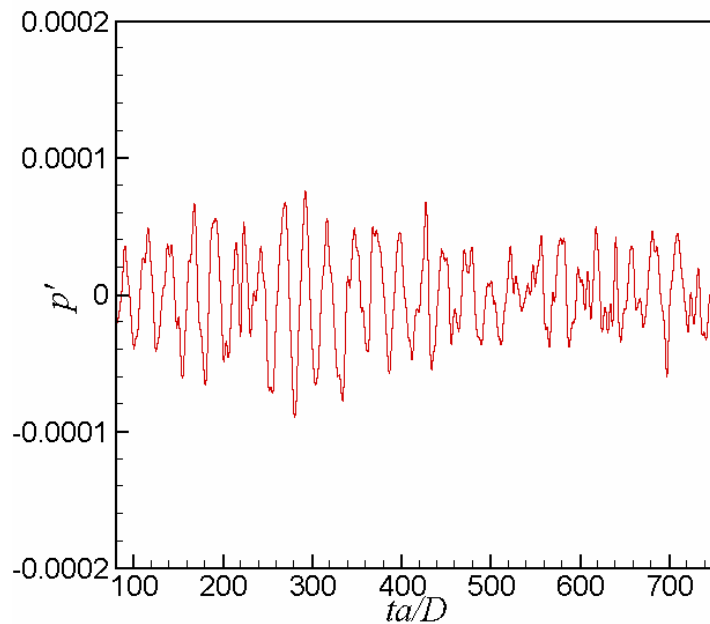
# Pressure fluctuation field

---

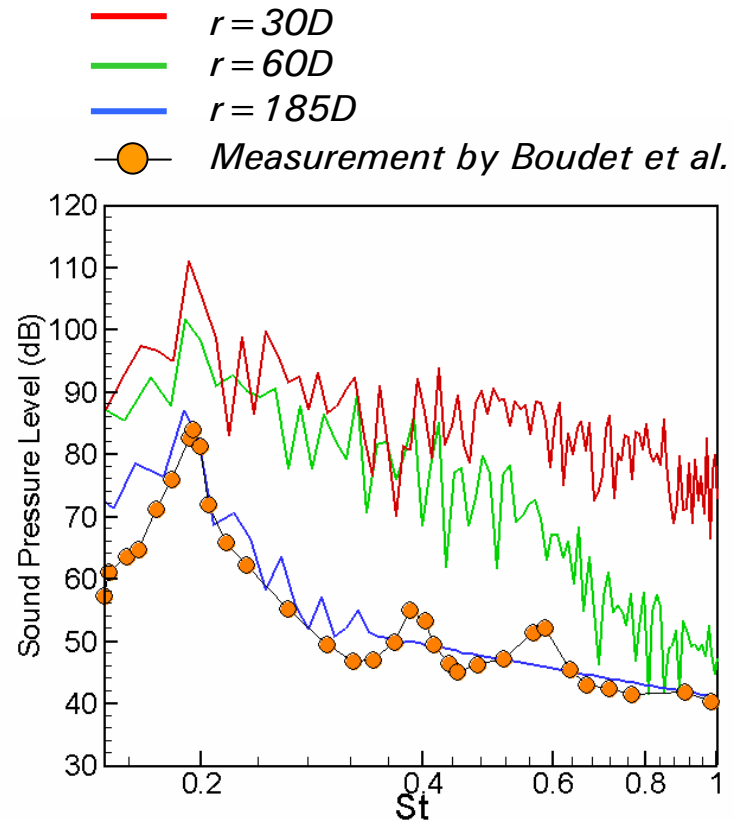




# Acoustic pressure signal & SPL spectrum



Time history of fluctuating pressure at  $r=30D$  directly above the cylinder



Frequency spectrum of acoustic pressure

*Kato et al. (if a spanwise coherence length,  $L_c \leq L_s$ )*  

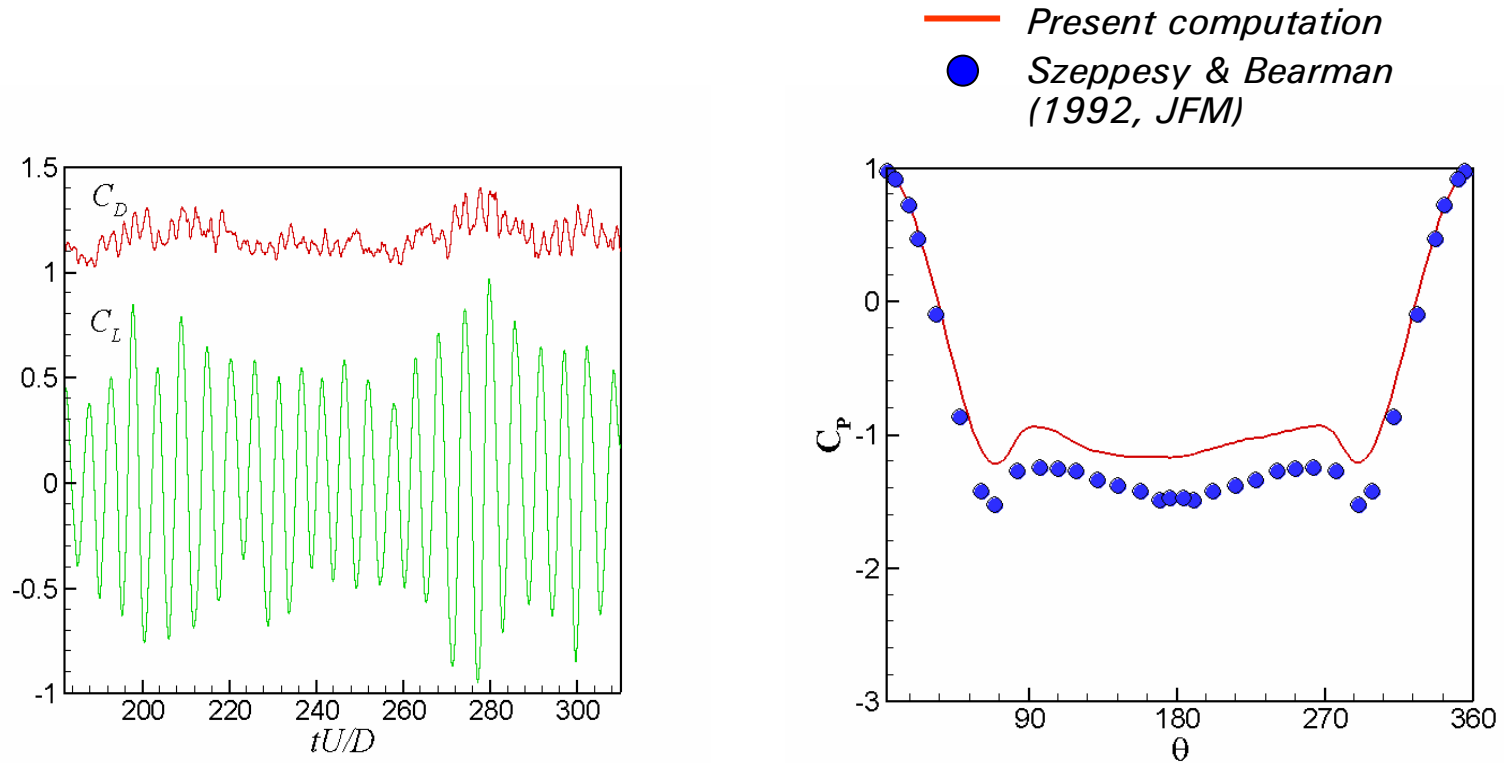
$$SPL_T = SPL_S + 10 \log(L_T / L_S) \text{ (dB)}$$



---

# Validation – *iLES* (Smagorinsky model)

---

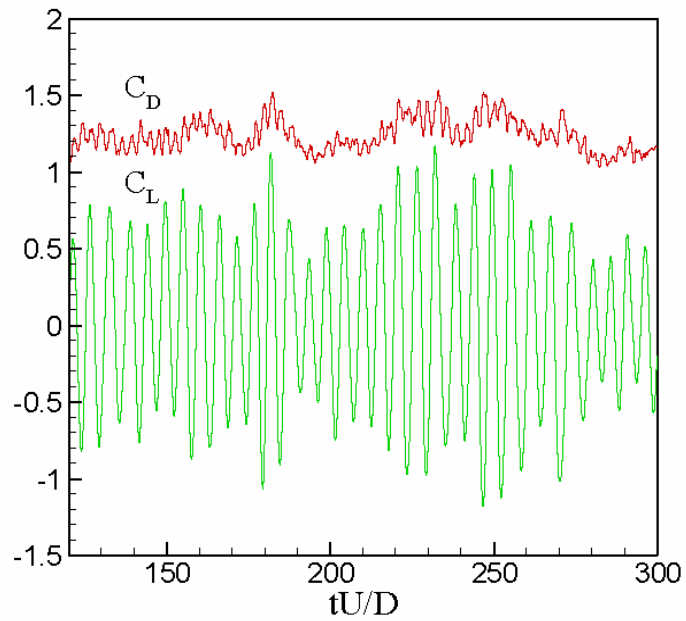


Time histories of drag and lift coefficients

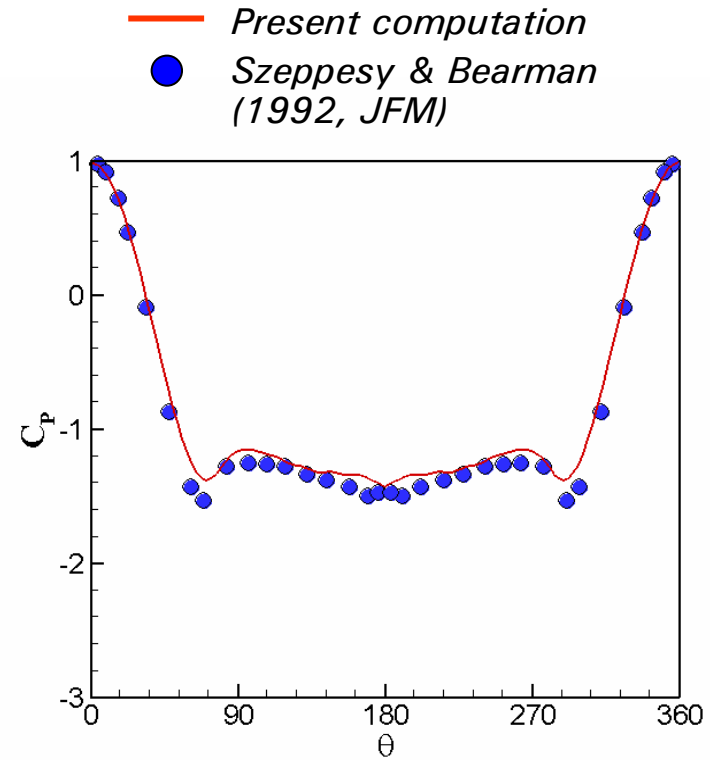
Mean pressure coefficient at the wall

# Validation – *iLES* (no model)

New Calc.



Time histories of drag and lift coefficients



Mean pressure coefficient at the wall



---

## Validation – Mean & Fluctuating data (no model)

---

*New Calc.*

	Present LES ( <i>No model</i> )	Experiment ( <i>Szepessy &amp; Bearman</i> )
$St$	<b>0.19</b> (0.19)*	0.19
$C_{D,avg}$	<b>1.24</b> (1.15)*	1.35
$C_{D' rms}$	<b>0.1</b> (0.067)*	0.16
$C_{L' rms}$	<b>0.54</b> (0.44)*	0.45-0.5

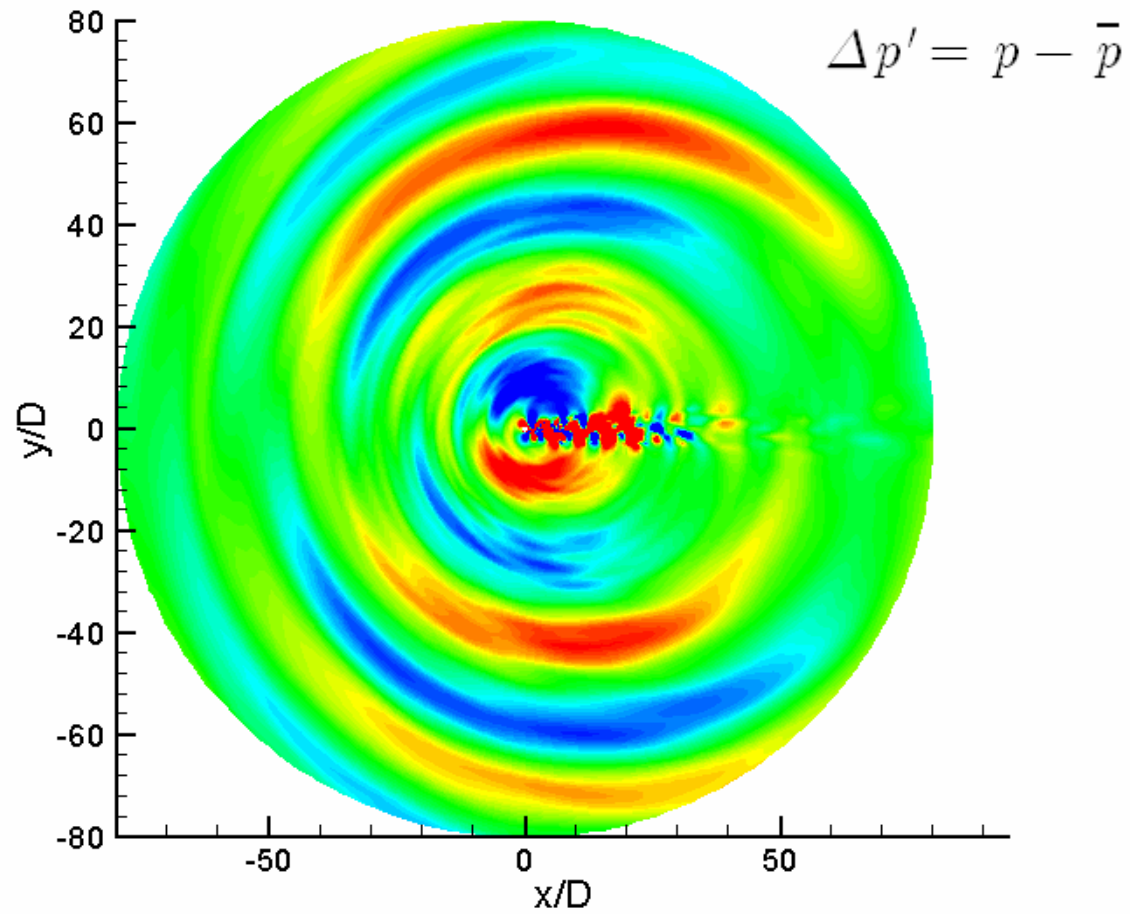


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# Pressure fluctuation field (no model)

---

New Calc.



---

---

## V. Phonation Aeroacoustics in Vocal Tract

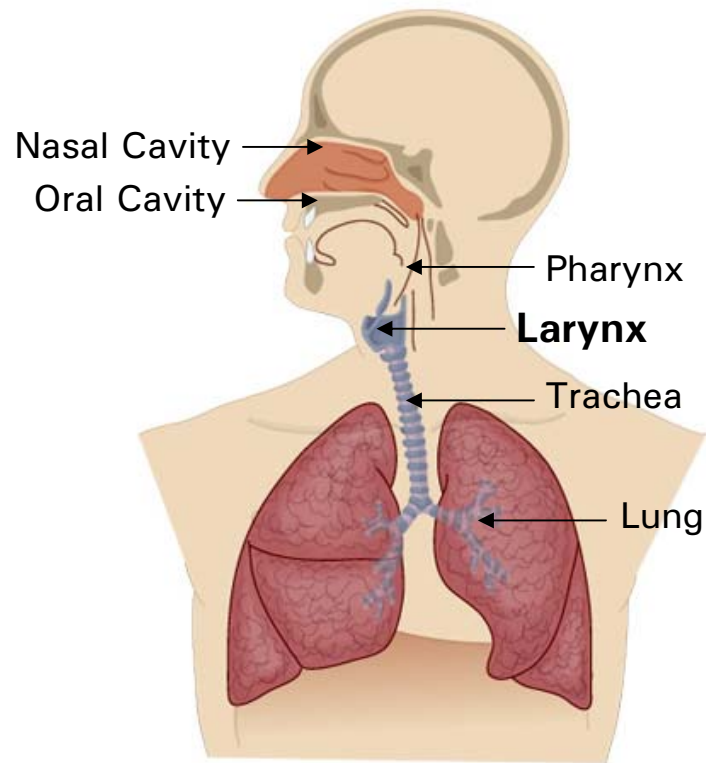


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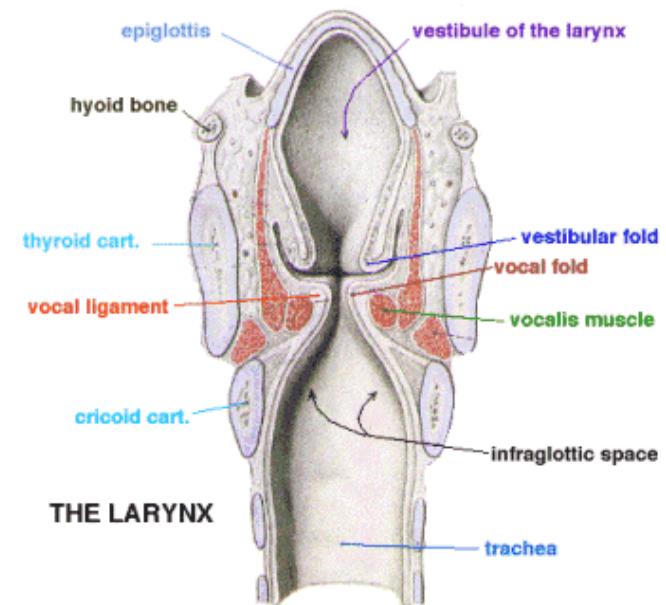
# Introduction

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## Review of Anatomy



Basic components of the airway system

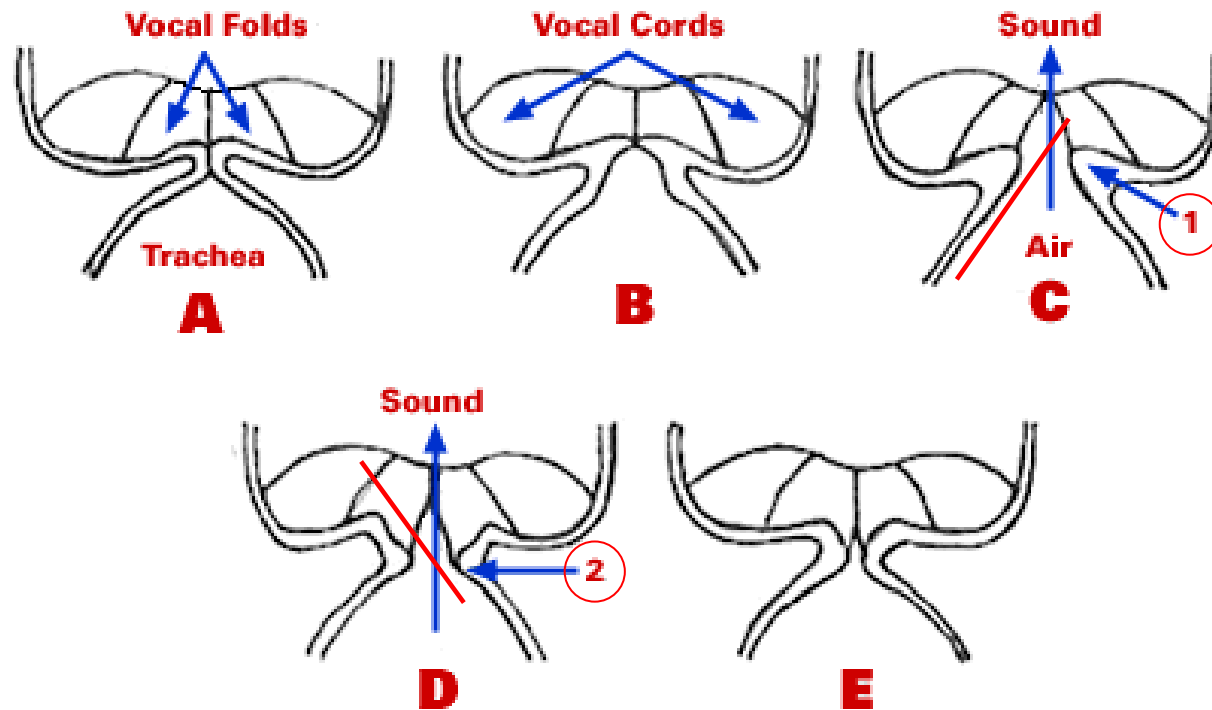


Coronal section through the larynx

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# Mechanism of phonation

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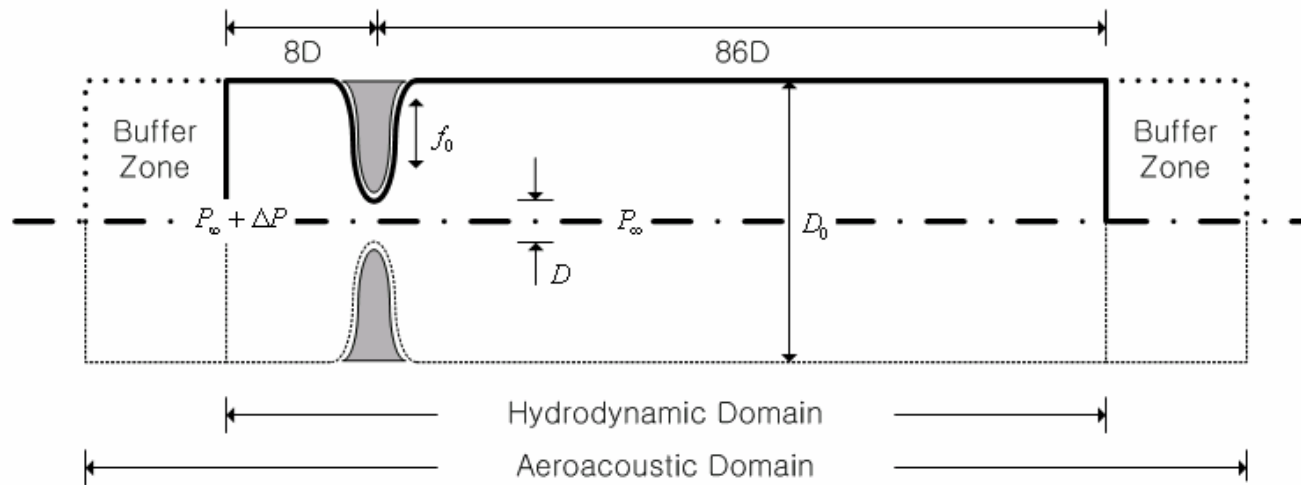


Self-sustained oscillations of the vocal folds → sound source

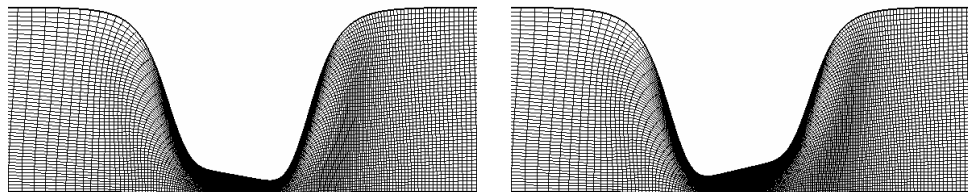




# Computational modeling (axi-symmetric)



$\Delta P$	$D_{min}$	$D_{max}$	$D_0$	$f_0$
832 Pa	0.8 mm	4.464 mm	5 $D_{max}$	94 Hz ( <i>male</i> )



Moving hydrodynamic grid (301x51)



---

## Governing equations

---

**Hydrodynamic:** axi-symmetric, incompressible N-S equations (**INS**) in moving coordinates,

$$\nabla \cdot \vec{U} = 0$$

$$\frac{\partial \vec{U}}{\partial t} + (\vec{U} \cdot \nabla) \vec{U} = -\frac{1}{\rho_0} \nabla P + \frac{\mu_0}{\rho_0} (\nabla^2 \vec{U})$$

**Acoustic:** axi-symmetric, perturbed compressible equations (**PCE**) in moving coordinates,

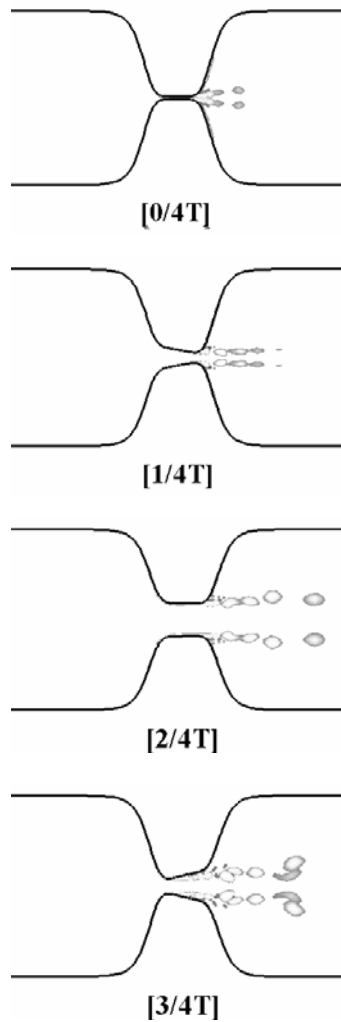
$$\frac{\partial \rho'}{\partial t} + (\vec{u} \cdot \nabla) \rho' + \rho (\nabla \cdot \vec{u}') = 0$$

$$\frac{\partial \vec{u}'}{\partial t} + (\vec{u} \cdot \nabla) \vec{u}' + (\vec{u}' \cdot \nabla) \vec{U} + \frac{\rho'}{\rho} \frac{D\vec{U}}{Dt} + \frac{1}{\rho} \nabla p' = \frac{\mu_0}{\rho} (\nabla \cdot \vec{f}')$$

$$\frac{\partial p'}{\partial t} + (\vec{u} \cdot \nabla) p' + \gamma p (\nabla \cdot \vec{u}') + (\vec{u}' \cdot \nabla) P = -\frac{DP}{Dt}$$



# Glottis transient profile during motions



Time dependent glottal radius (Zhao et al, 2002)

$$r_w(x,t) = \frac{D_0 - D_{\min}}{4} \tanh(s) + \frac{D_0 + D_{\min}}{4} + \frac{1}{2} [1 - \tanh(s)] \times [\beta_1(x + cD_{\max}) - \beta_2(x - cD_{\max})]$$

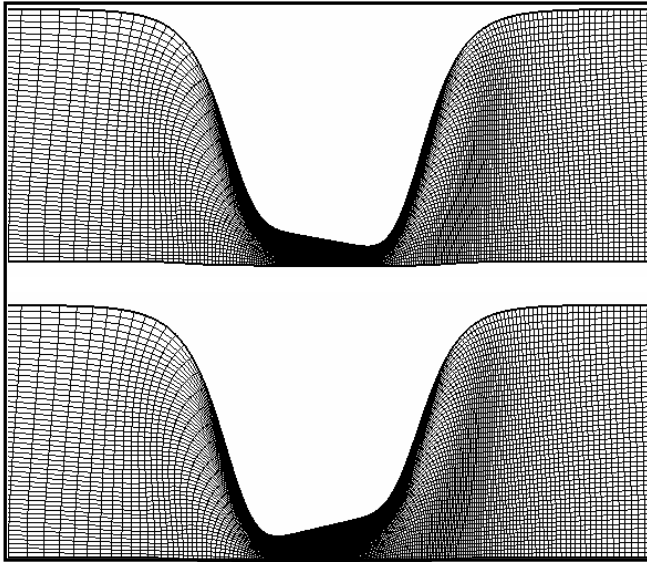
$$s = \frac{b|x|}{D_{\max}} - \frac{bD_{\max}}{|x|}, \quad \beta_2(t) = \beta_1\left(t + \frac{T}{9}\right), \quad b = 1.4 \text{ and } c = 0.42,$$

$$\beta_1(t) = \begin{cases} 0, & \left(t \leq \frac{1}{9}T\right) \\ 0.244 \left[1 - \cos\left\{\frac{9\pi}{4}\left(\frac{t}{T} - \frac{1}{9}\right)\right\}\right], & \left(\frac{1}{9}T < t \leq \frac{5}{9}T\right) \\ 0.488, & \left(\frac{5}{9}T < t \leq \frac{6}{9}T\right) \\ 0.244 \left[1 + \cos\left\{3\pi\left(\frac{t}{T} - \frac{6}{9}\right)\right\}\right], & \left(\frac{6}{9}T < t \leq T\right) \end{cases}$$

---

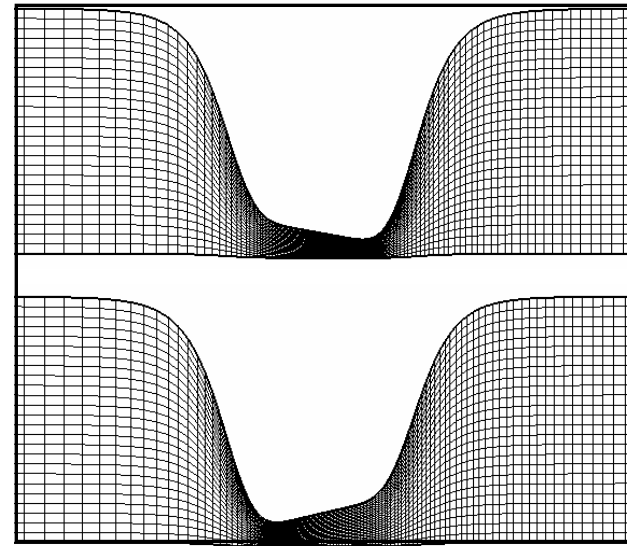
## Moving grids for *INS* & *PCE*

---



Moving hydrodynamic grid (301x51)

O

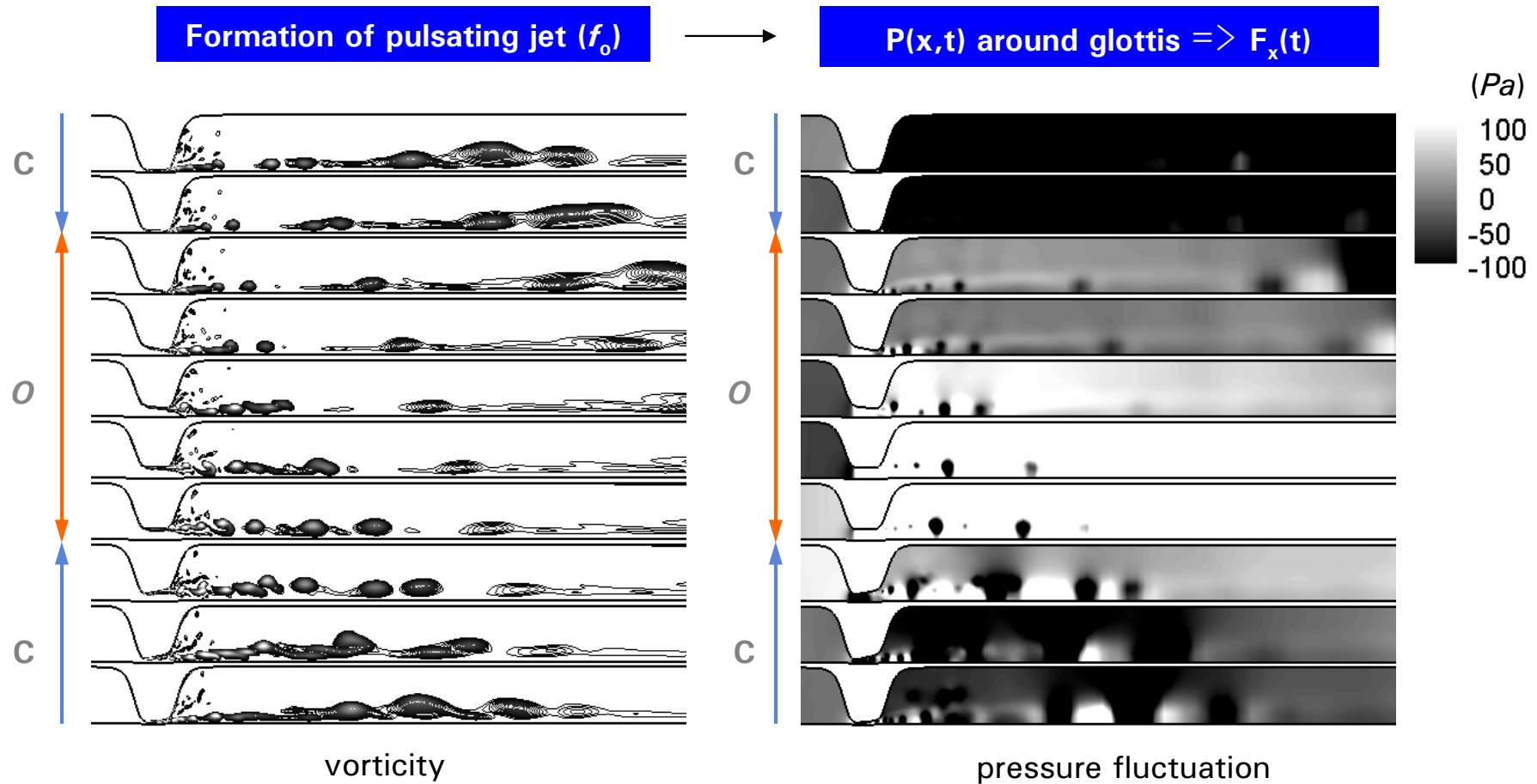


C

Moving acoustic grid (251x26)



# Hydrodynamic & pressure fluctuation fields

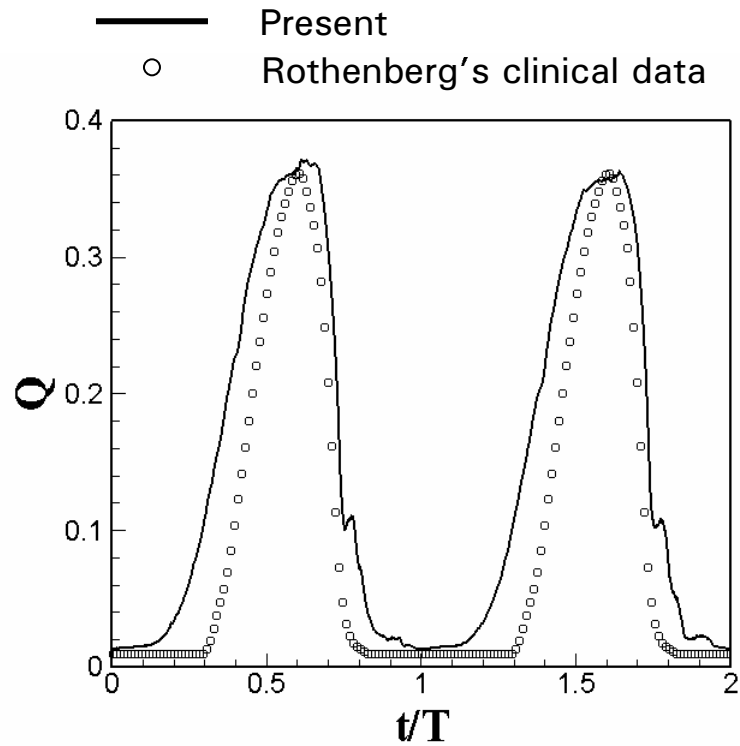


Among Monopole, Dipole, & Quadrupole, Dipole sound dominates in vocal phonation by an (oscillating) piston effect.

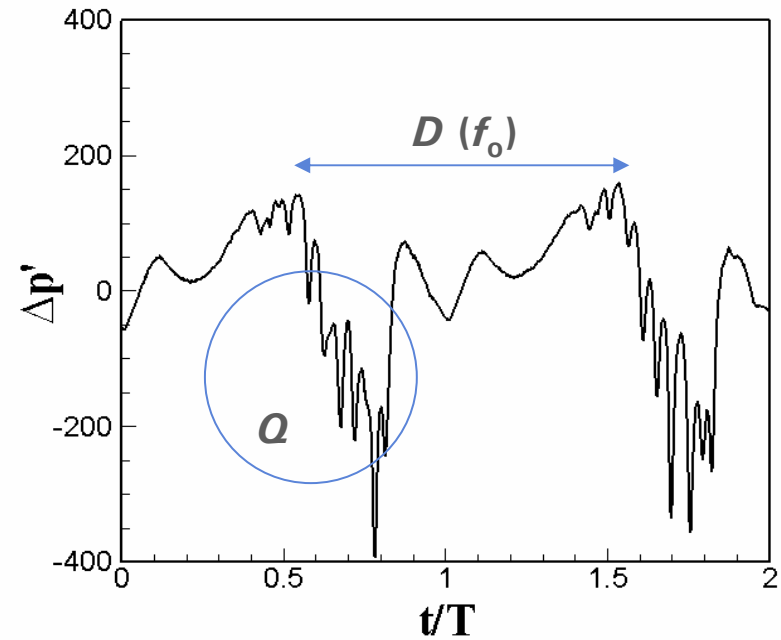


# 1. Validation

$\Delta P = 8.1 \text{ cmH}_2\text{O}$ ,  $f_o = 94 \text{ Hz}$



Volume flow rate (l/s)

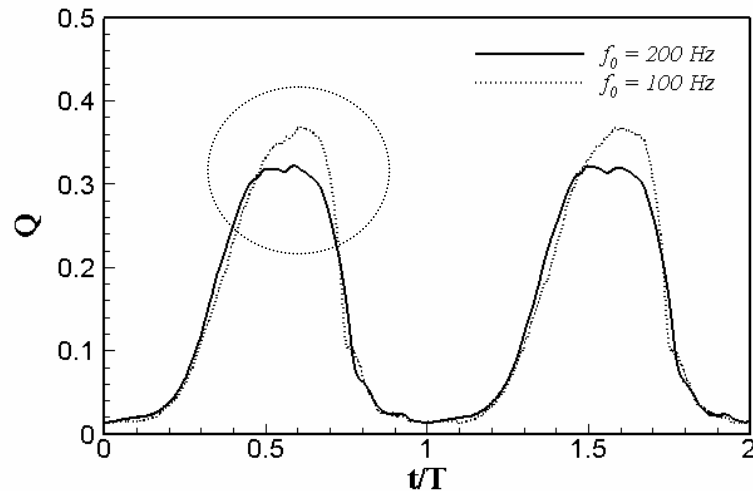


Acoustic pressure (Pa) at 10Dmax downstream

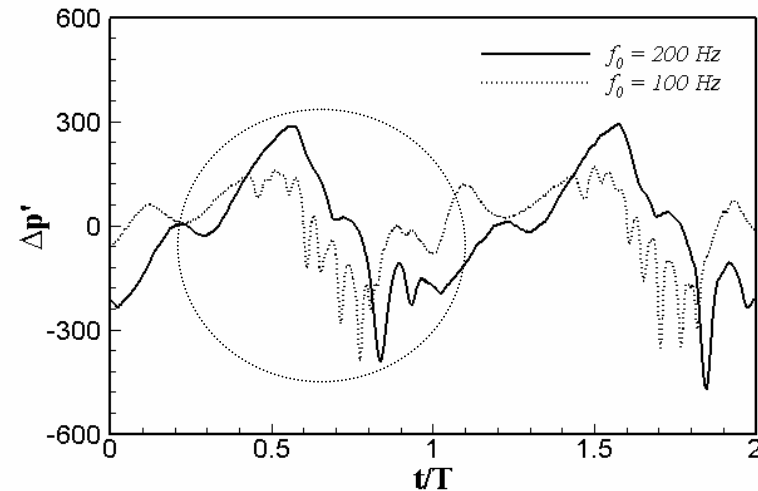
$SPL = 110 \text{ dB}$



## 2. Effect of fundamental frequency (male & female)



Volume flow rate (l/s) through the glottis



Acoustic pressure (Pa) at  $10D_{max}$  downstream

Variation of fundamental frequency  $\rightarrow$  change of Jet Reynolds No.  
 $\rightarrow$  change of Jet flow characteristics

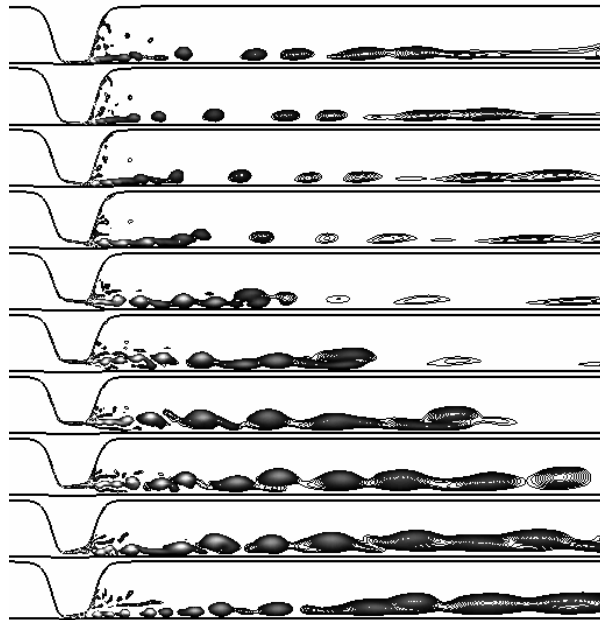
High  $f_0$  generates fluty sound, while low  $f_0$  makes sound brassy



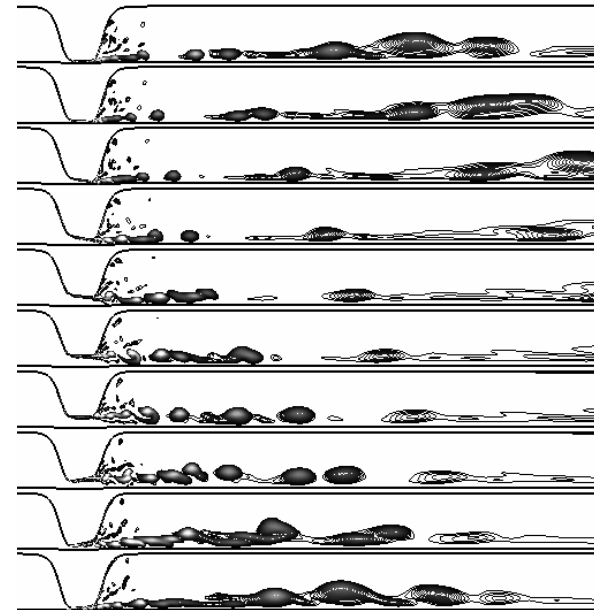
---

## Effect of fundamental frequency (male & female)

---



$f_0 = 94\text{Hz}$  (male)



$f_0 = 200\text{Hz}$  (female)

$$Re_f < Re_m$$

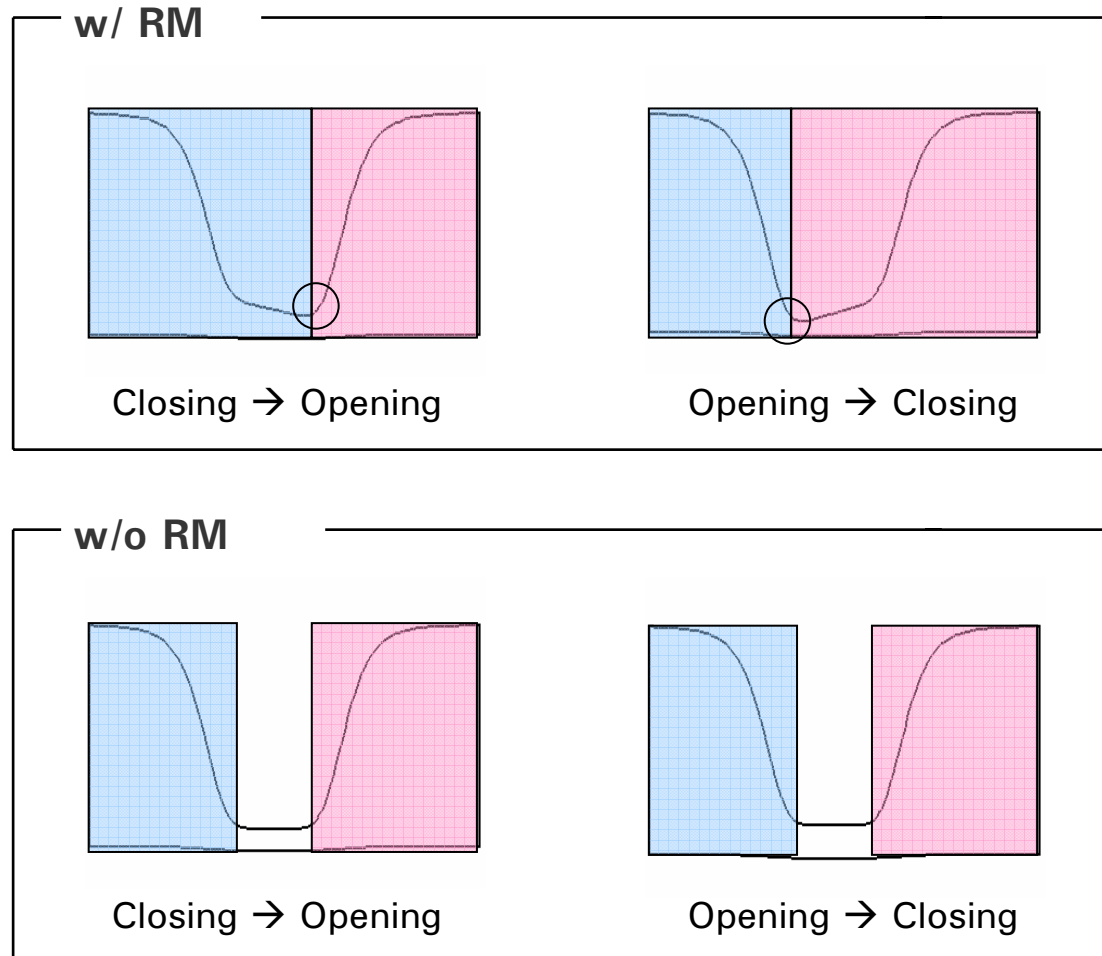




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### 3. Effect of rotational motion of Glottis

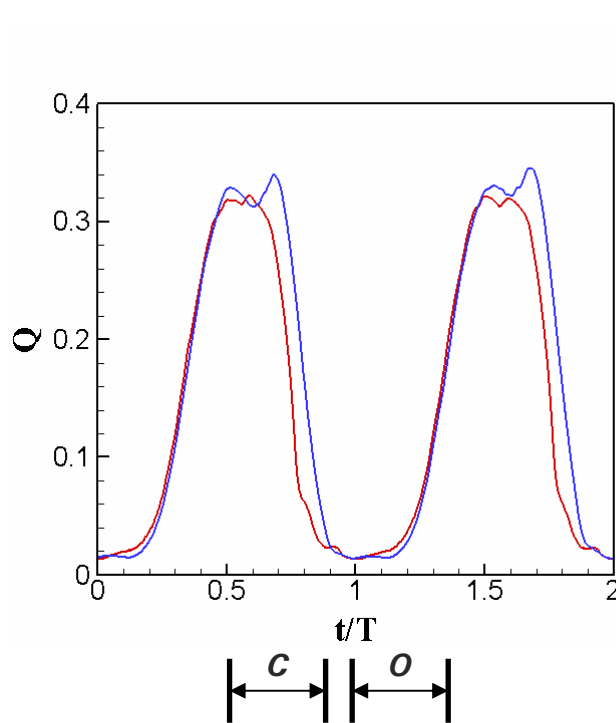
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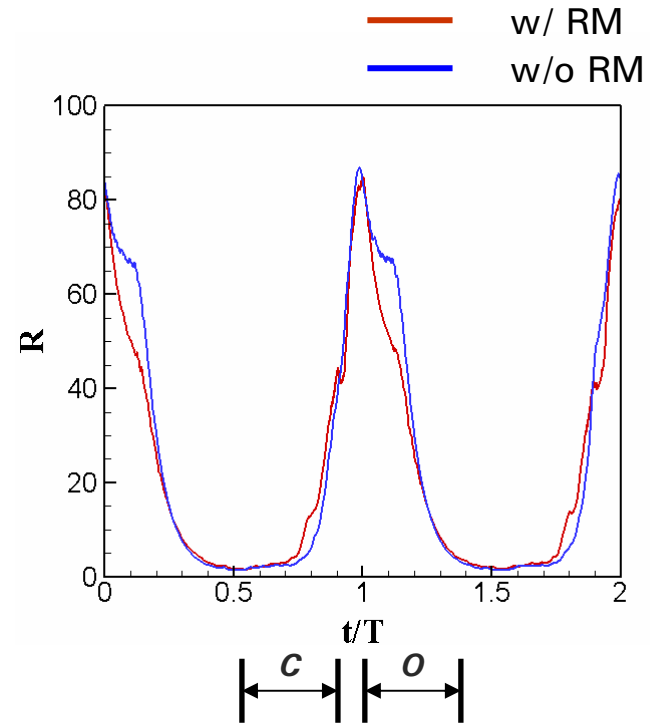
# Effect of rotational motion

$f_0 = 200\text{Hz}$ ,  $\Delta P = 8.1\text{cmH}_2\text{O}$

Glottal Resistance:  $R = \Delta P / Q$  (Van den Berg, 1957)



Volume flow rate (l/s) through the glottis



Periodic variation of glottal impedance ( $\text{kPa}\cdot\text{s/l}$ )



---

## Effects of rotational motion

---

For a sound generator (glottis) to be mechanically efficient, compression waves should be generated with minimum glottal resistance at opening and rarefaction waves with maximum glottal resistance at closure.

With rotational motion,  $R$  is clearly maximum at *C-stage* and minimum at *O-stage*, which means that *rotational motion* of glottis is related to the effectiveness of rarefaction & compression effects.



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## Conclusions and future work

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- Understanding of sound generation mechanisms and characteristics.
- Computational results agree well with the inverse filtered data by Rothenberg.
- Fundamental frequency of glottis has effect on voice quality.
- **Rotational motion** of glottis is associated with the **efficiency** of vocal folds motion via **glottis impedance**.
- In the future, we hope to include the fluid-structure-interaction (**FSI**) to replicate the **self-sustained motions** of glottis and then the **acoustic coupling**.



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**Questions & Answers.**

Thank you for your attentions.

